

# A METHOD FOR THE EFFICIENT MODELLING OF DELAMINATION IN LARGE STRUCTURES

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AMADE Day  
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UdG

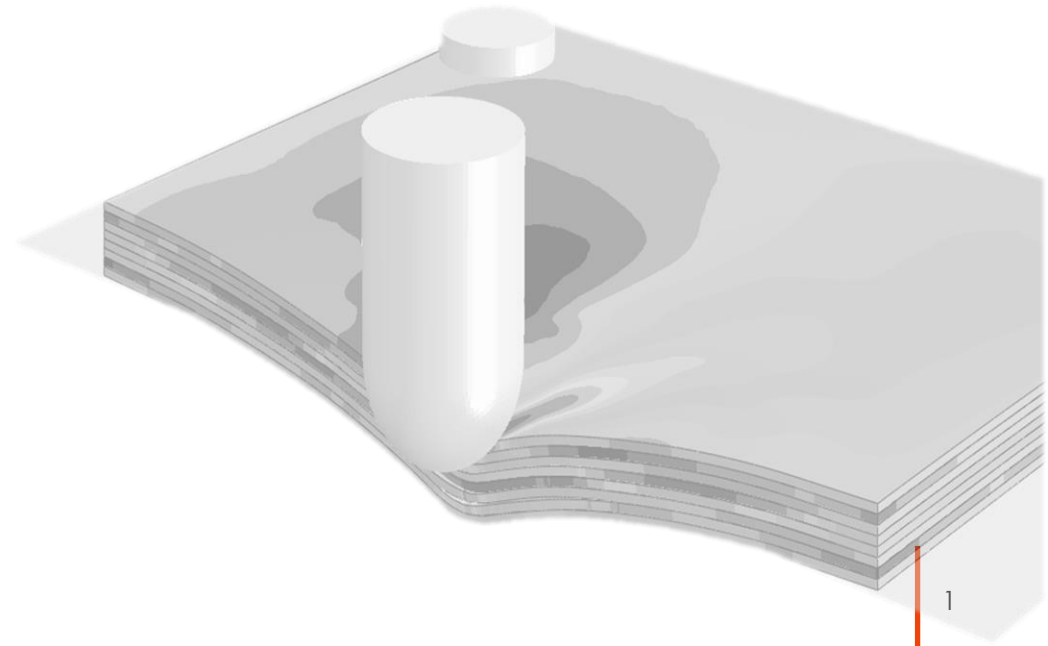


**AMADE**  
ANALYSIS AND ADVANCED MATERIALS  
FOR STRUCTURAL DESIGN

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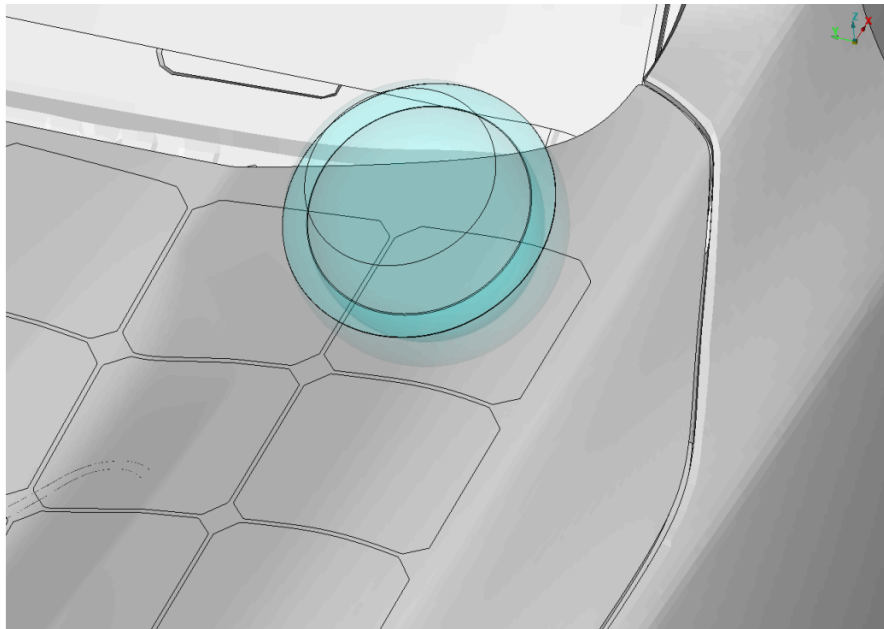


**CHALMERS**  
UNIVERSITY OF TECHNOLOGY



# Introduction

Need for the modelling of delamination **in large structures**, an example:



**Headform impact** (pedestrian safety):

Energy: 212 Joules

Mass: 4.5 kg

Velocity: 9.7 ms<sup>-1</sup>

Diameter: 165 mm

About **50 positions** to model on the hood.

Simulation time (26cpus):

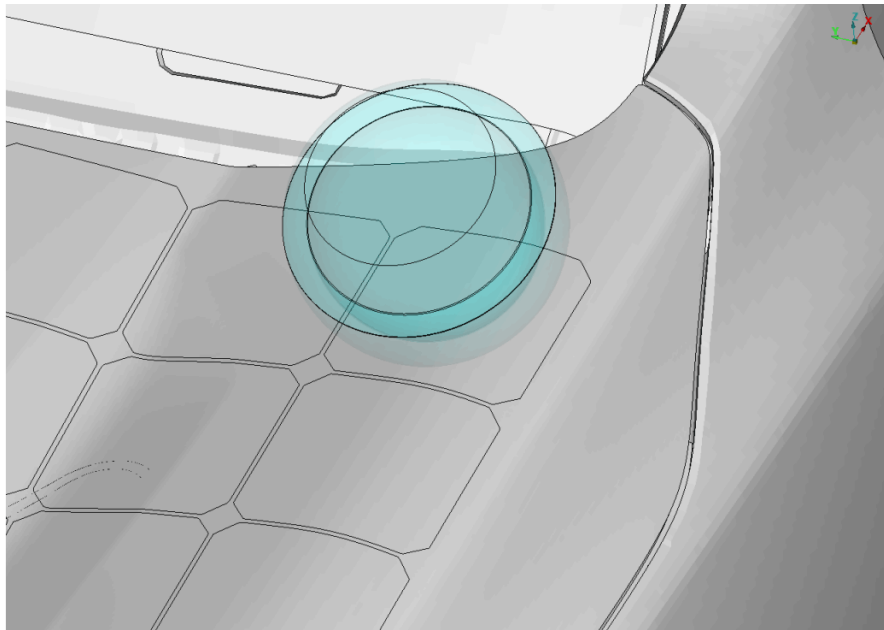
Steel hood: 22 minutes

Composite hood: 8 hours



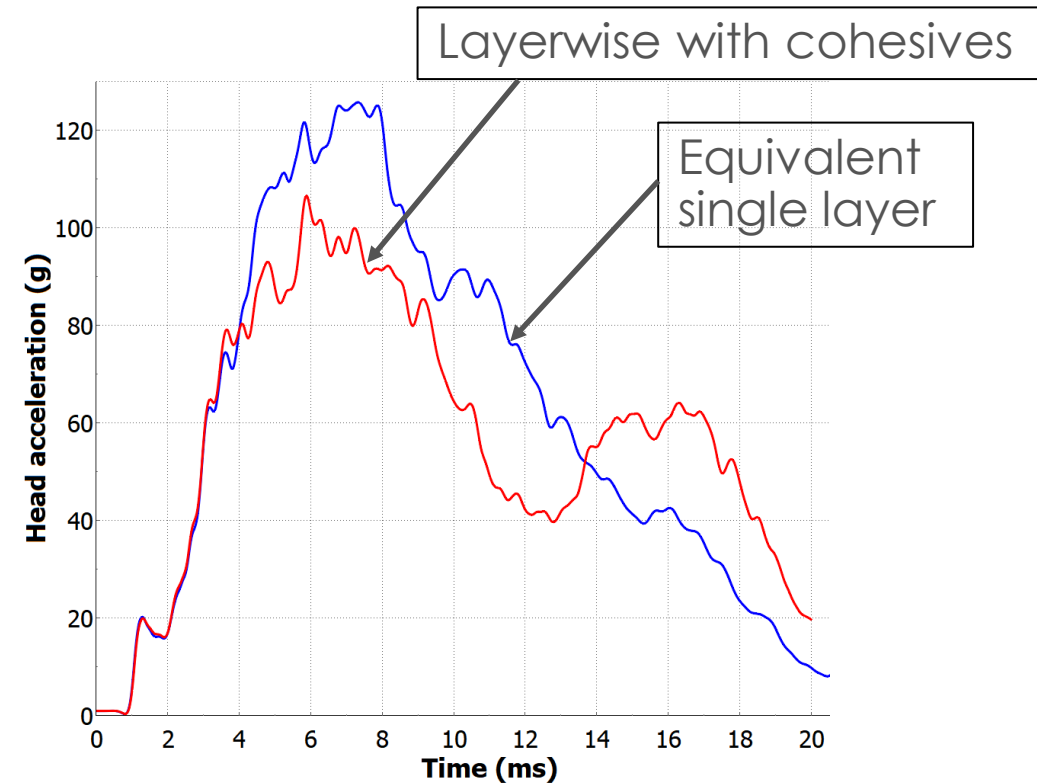
# Introduction

Need for the modelling of delamination in large structures, an example:



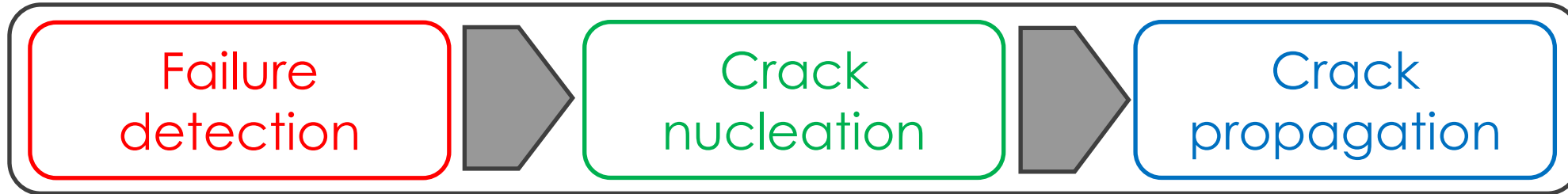
Simulation time (26cpus):

Steel hood:	22 minutes	<span style="color: red; font-size: 2em;">↓ x 22</span>
Composite hood:	8 hours	



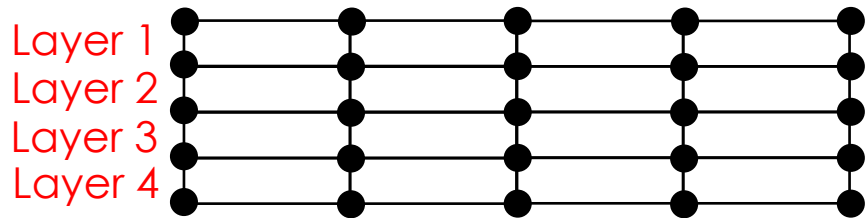
The “**Head injury criterion**” is halved !

# Delamination modelling (State of the art)

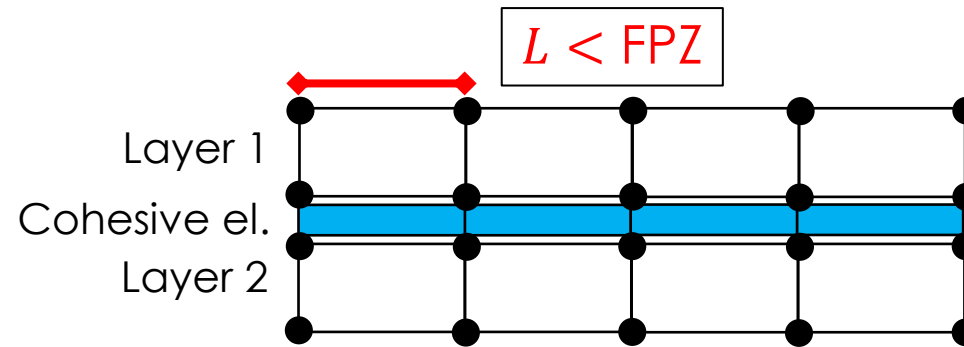


Need: out-of-plane stress  
 Layerwise modelling

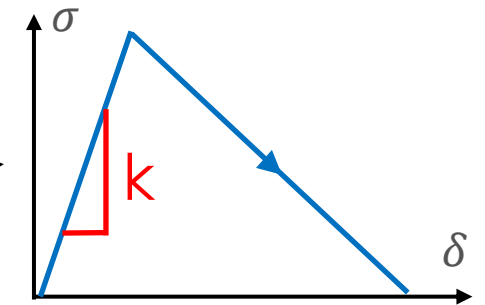
Cohesive Zone Modelling



**Thickness refinement**  
 $\times N_{layers}$



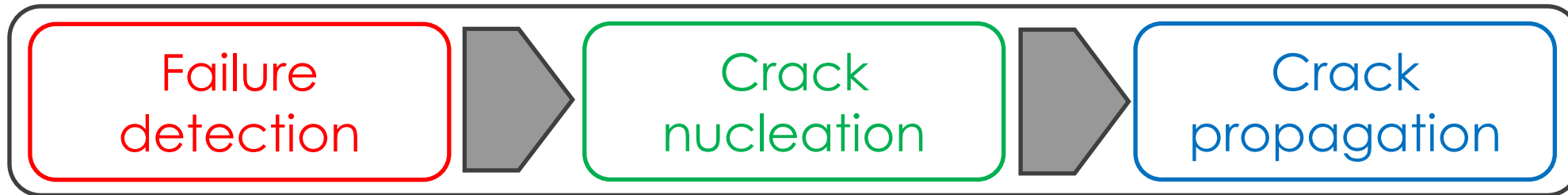
**In-plane refinement**  
 $\times 25$   
 (e.g. 1 mm vs 5 mm)



**Time increments**  
 $\times 10$   
 (e.g.  $0.5 \times 10^{-7}$  s vs  $0.5 \times 10^{-6}$  s)

For a 10-layer laminate, the computational cost is increased by a factor  $\sim 2500$  !!

## Present modelling strategy

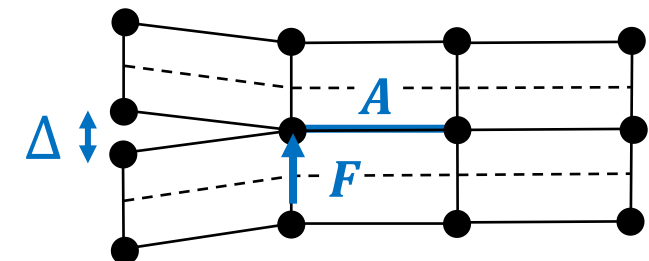
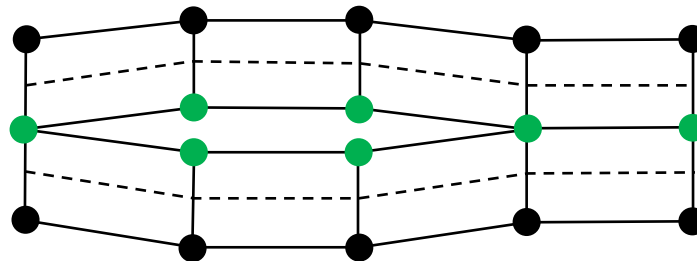
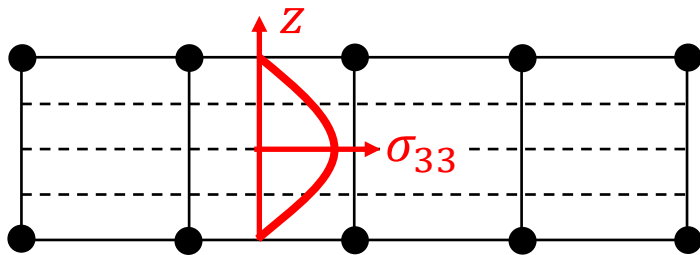


Equivalent single layer  
Stress Recovery method

Enrichment  
Adaptive modelling

ERR based propagation  
VCCT+cohesive

$\sigma_{13}, \sigma_{23}, \sigma_{33}$

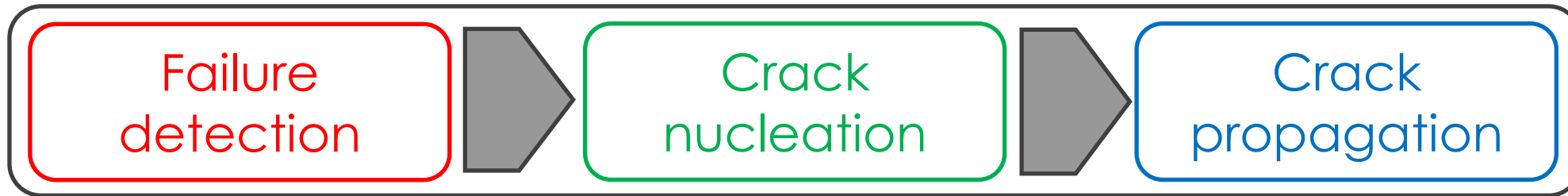


$Fl(\sigma_{13}, \sigma_{23}, \sigma_{33}) > \text{limit}$   
 ↳ Enrichment

● Extra node

$ERR(\Delta, F, A) > G_c \longrightarrow \text{propagation}$

## Present modelling strategy



Equivalent single layer  
Stress Recovery method

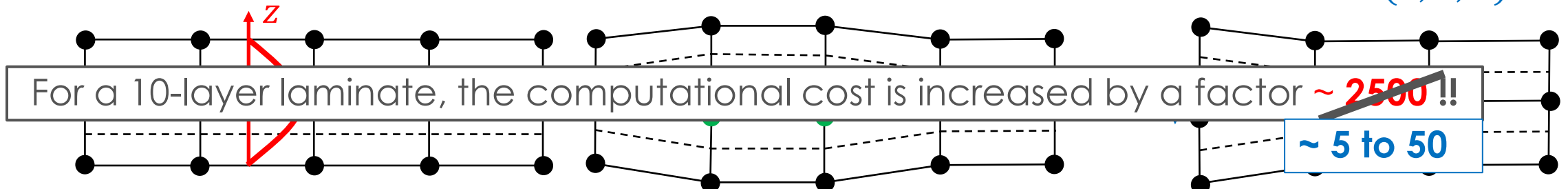
Enrichment  
Adaptive modelling

ERR based propagation  
VCCT+cohesive

$$\sigma_{13}, \sigma_{23}, \sigma_{33}$$

● Extra node

$$ERR(\Delta, F, A) > G_c$$



For a 10-layer laminate, the computational cost is increased by a factor ~~~ 2500 !!~~

**~ 5 to 50**

**Thickness refinement**

~~x  $N_{layers}$~~

**x 1** (no failure)  
**x  $N_{layers}/2$**   
(progressive failure\*)

**In-plane refinement**

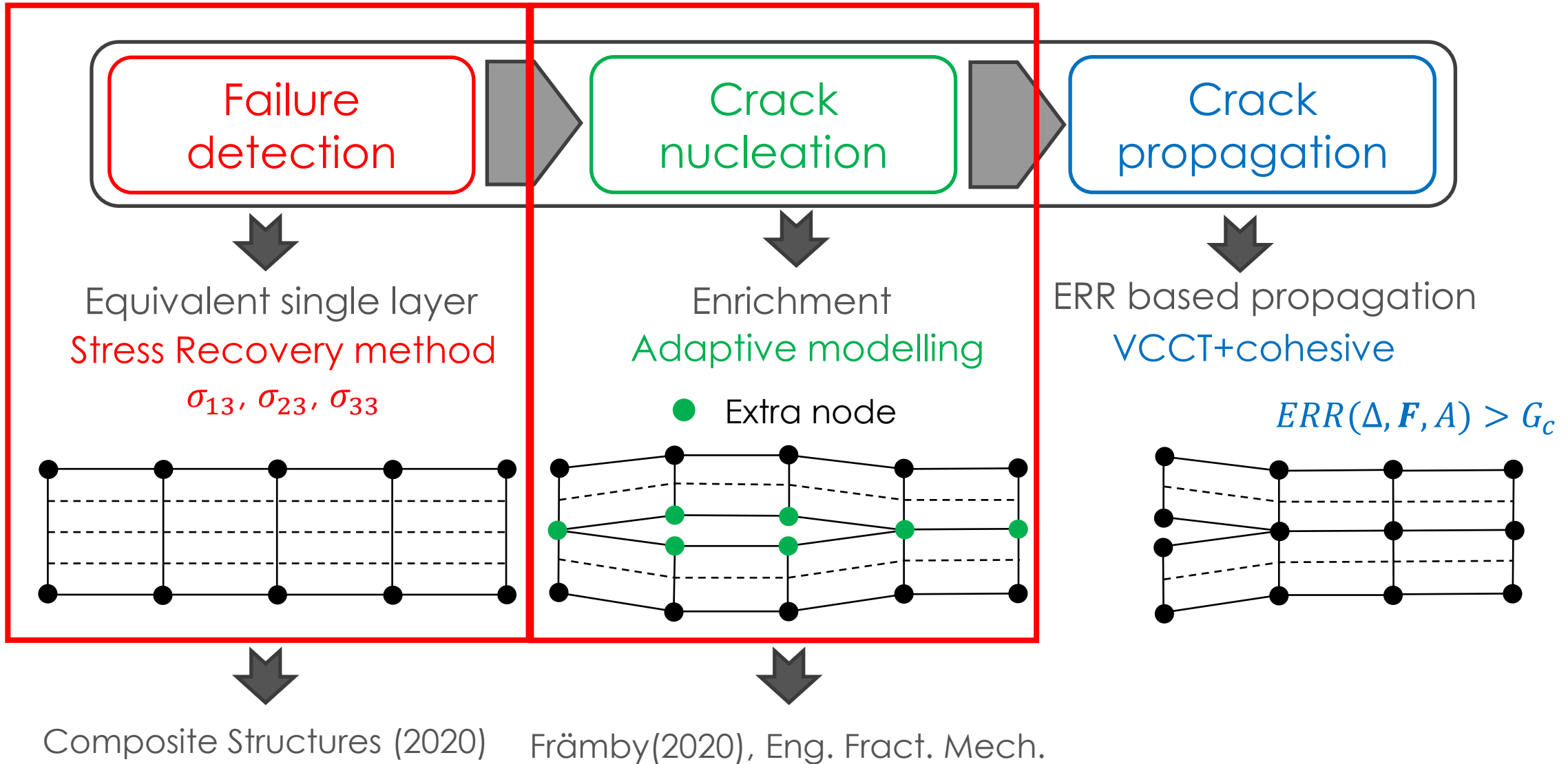
~~x 25~~  
(1 mm)

**x 1** (5 mm)  
**x 4** (2.5 mm)

**Time increments**

~~x 10~~  
( $0.5 \times 10^{-7}$  s)

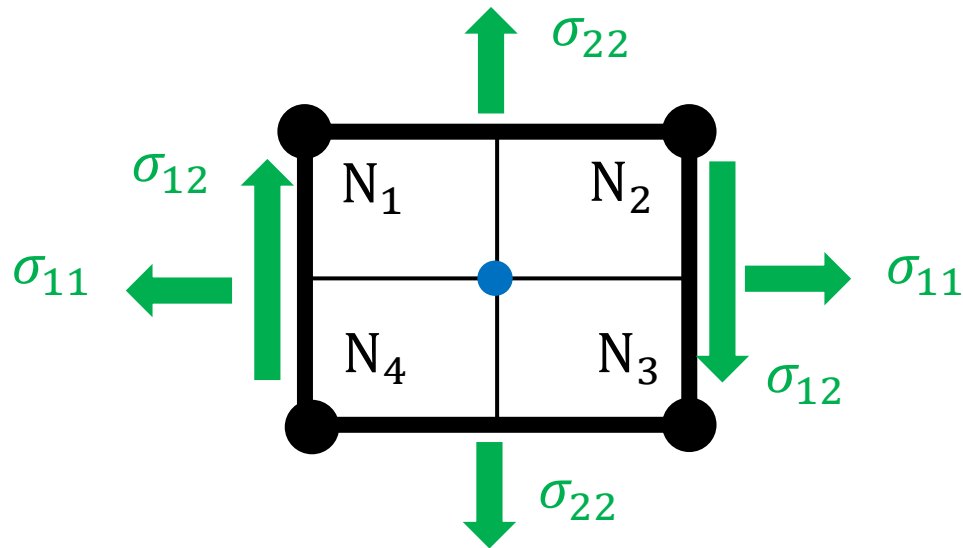
**x 5** ( $1.0 \times 10^{-7}$  s)



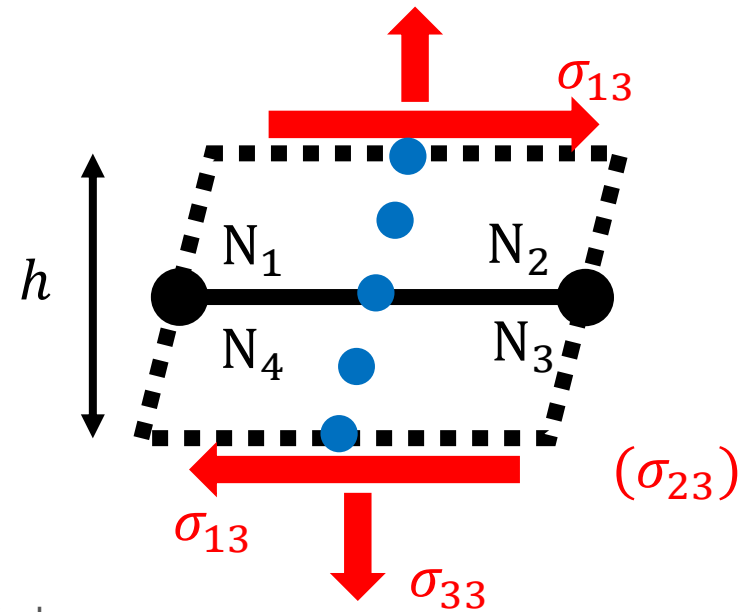
# Stress Recovery Basics

Linear Shell elements (ESL based on FSDT) present:

**Good quality** of in-plane stresses



**Poor quality** of transverse stresses



● Integration points



# Stress Recovery Basics

Linear Shell elements (ESL based on FSDT) present:

Several authors have proposed using the **stress equilibrium equations** :

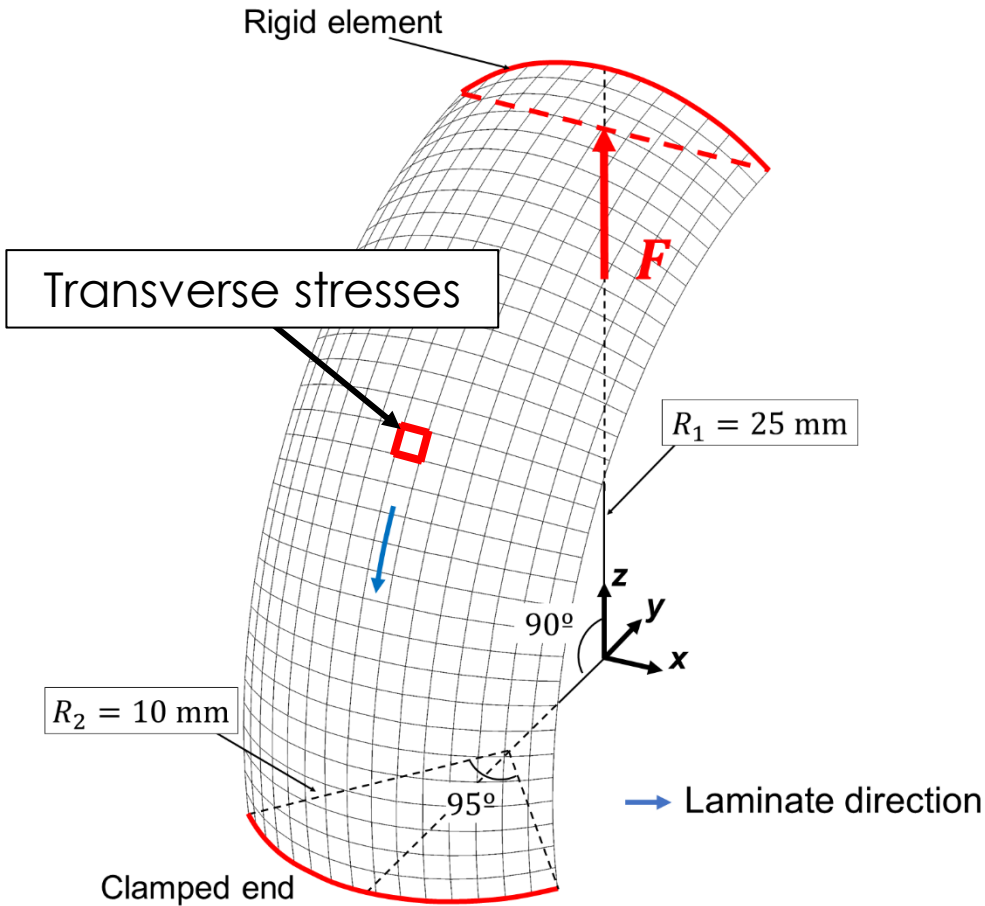
$$\begin{aligned} \sigma_{11,1} + \sigma_{12,2} + \sigma_{13,3} &= 0 \\ \sigma_{13,1} + \sigma_{23,2} + \sigma_{33,3} &= 0 \end{aligned} \quad \longrightarrow \quad \begin{aligned} \sigma_{13} &= - \int_{-\frac{h}{2}}^z \sigma_{11,1} + \sigma_{12,2} dz + C_1 \\ \sigma_{33} &= \iint_{-\frac{h}{2}}^z \sigma_{11,11} + \sigma_{22,22} + 2\sigma_{12,12} dz dz + C_3 \end{aligned}$$

These equations have been generalized for **arbitrarily curved geometries**.

An **efficient formulation**, based on the shell **forces and moments** is used.

$\sigma_{13}$ ,  $\sigma_{23}$  and  $\sigma_{33}$  are recovered in a post-processing step.

# Doubly-curved example: Bending



▣ Laminate:

$[(0; 90; 45; -45)_2; -45; 45; 90; 0; -45; 45; 0_8; 90; 0]$

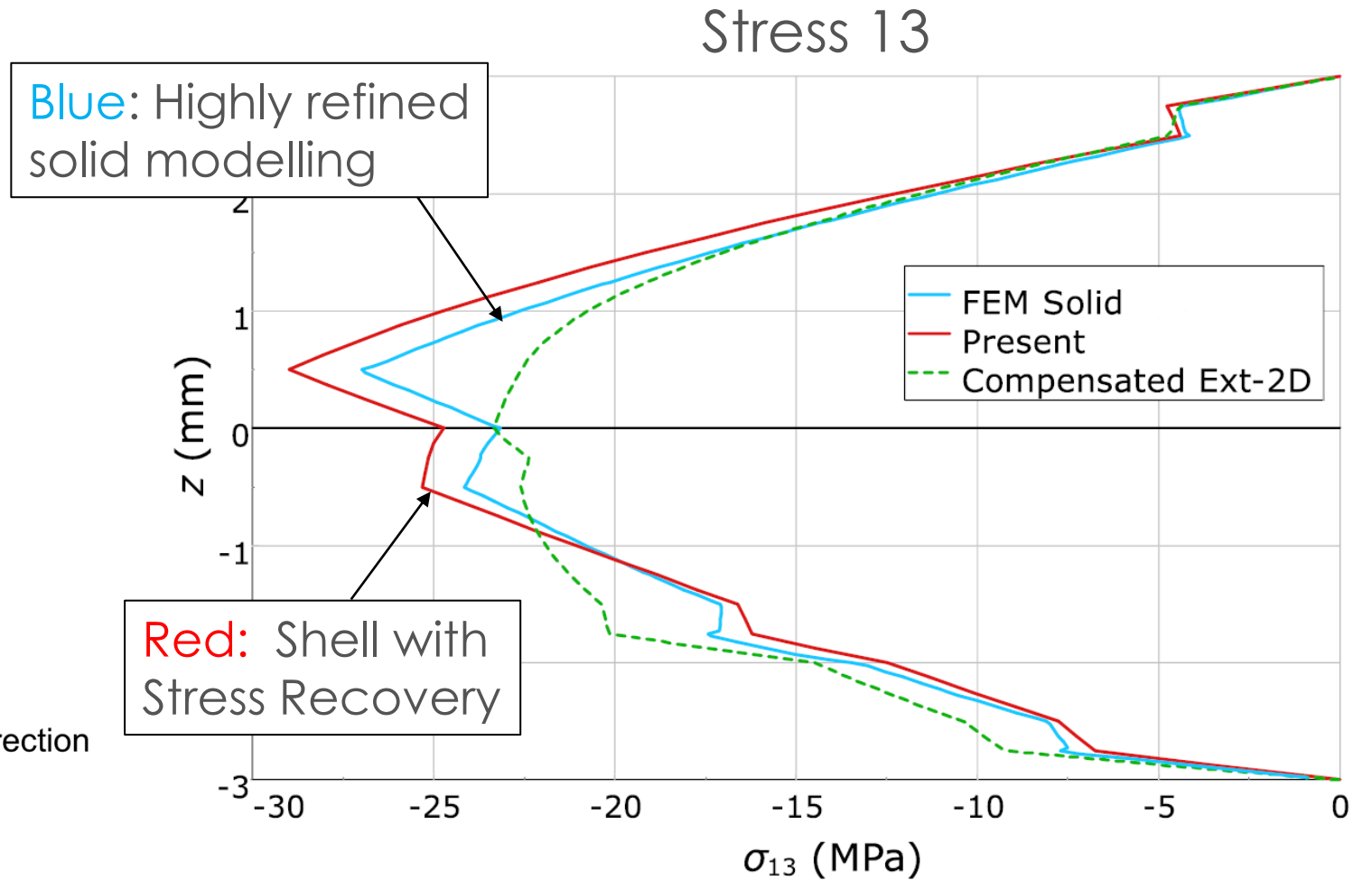
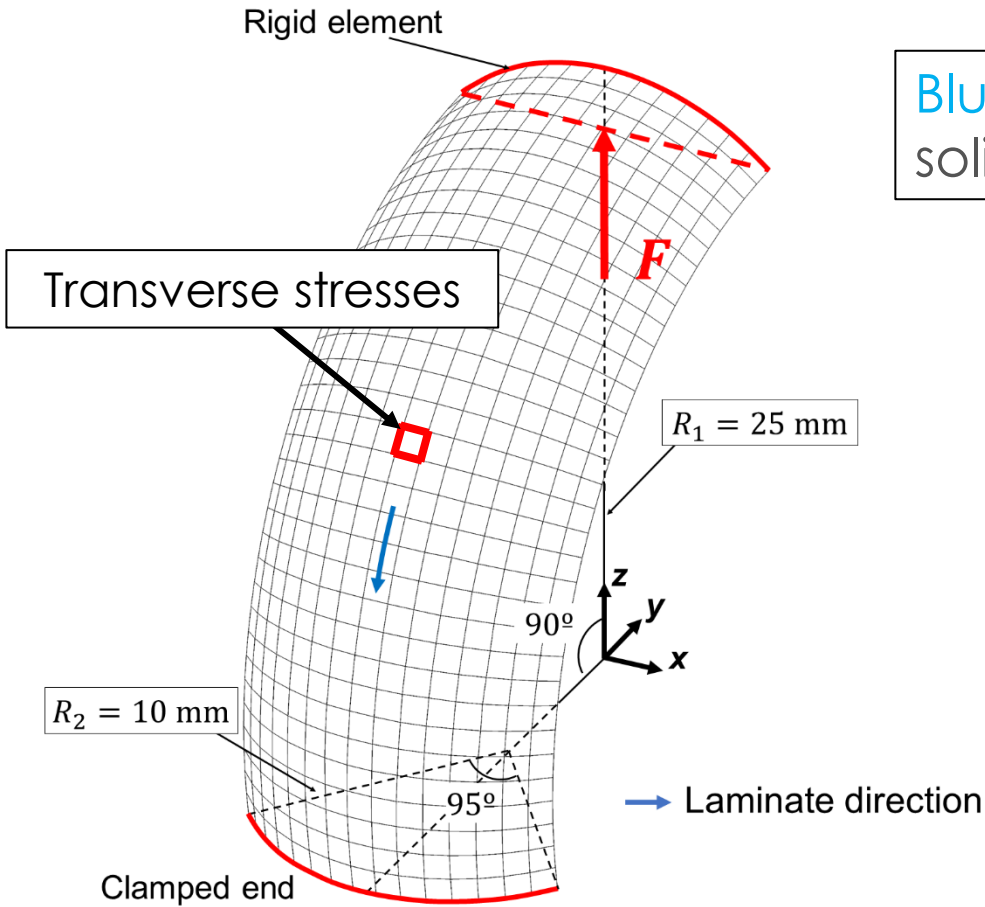
➤ Layer thickness : 0.25 mm

➤ Total thickness : 6 mm

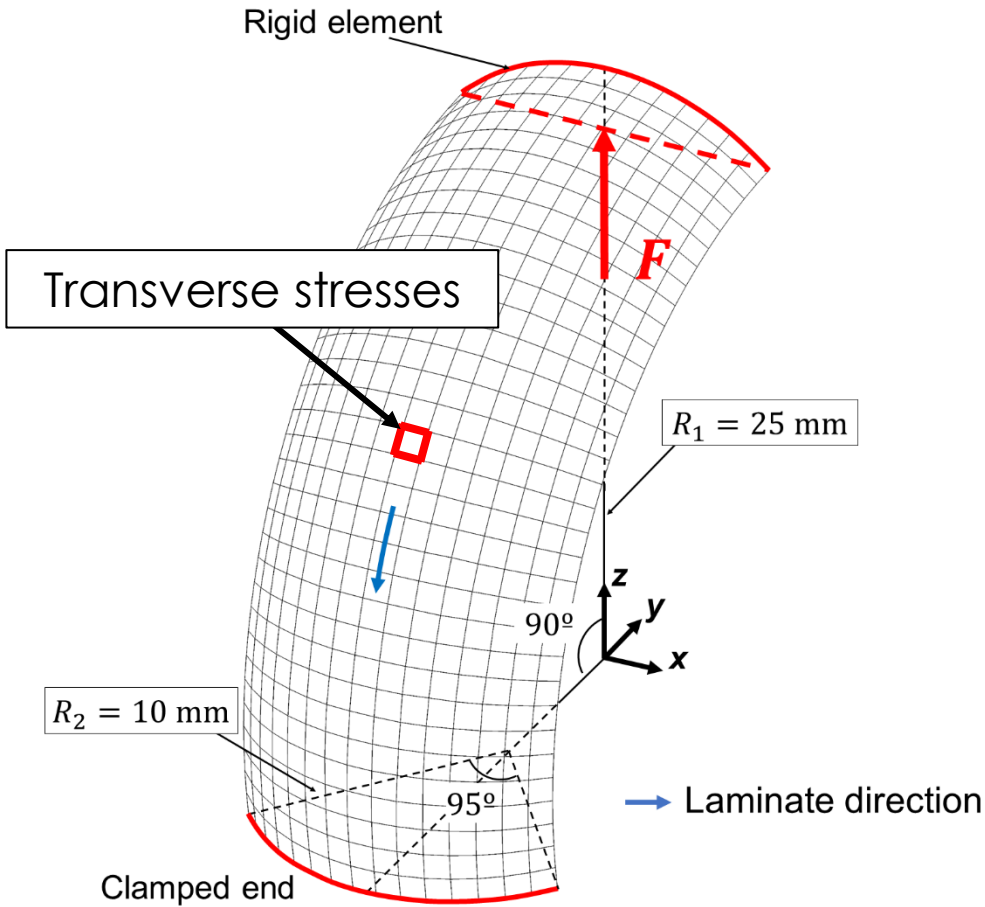
▣ Prototype material:

$E_{11}$	$E_{22} = E_{33}$	$G_{12} = G_{13}$
100 GPa	10 GPa	5 GPa
$G_{23}$	$\nu_{12} = \nu_{13} = \nu_{23}$	
4 GPa	0.25	

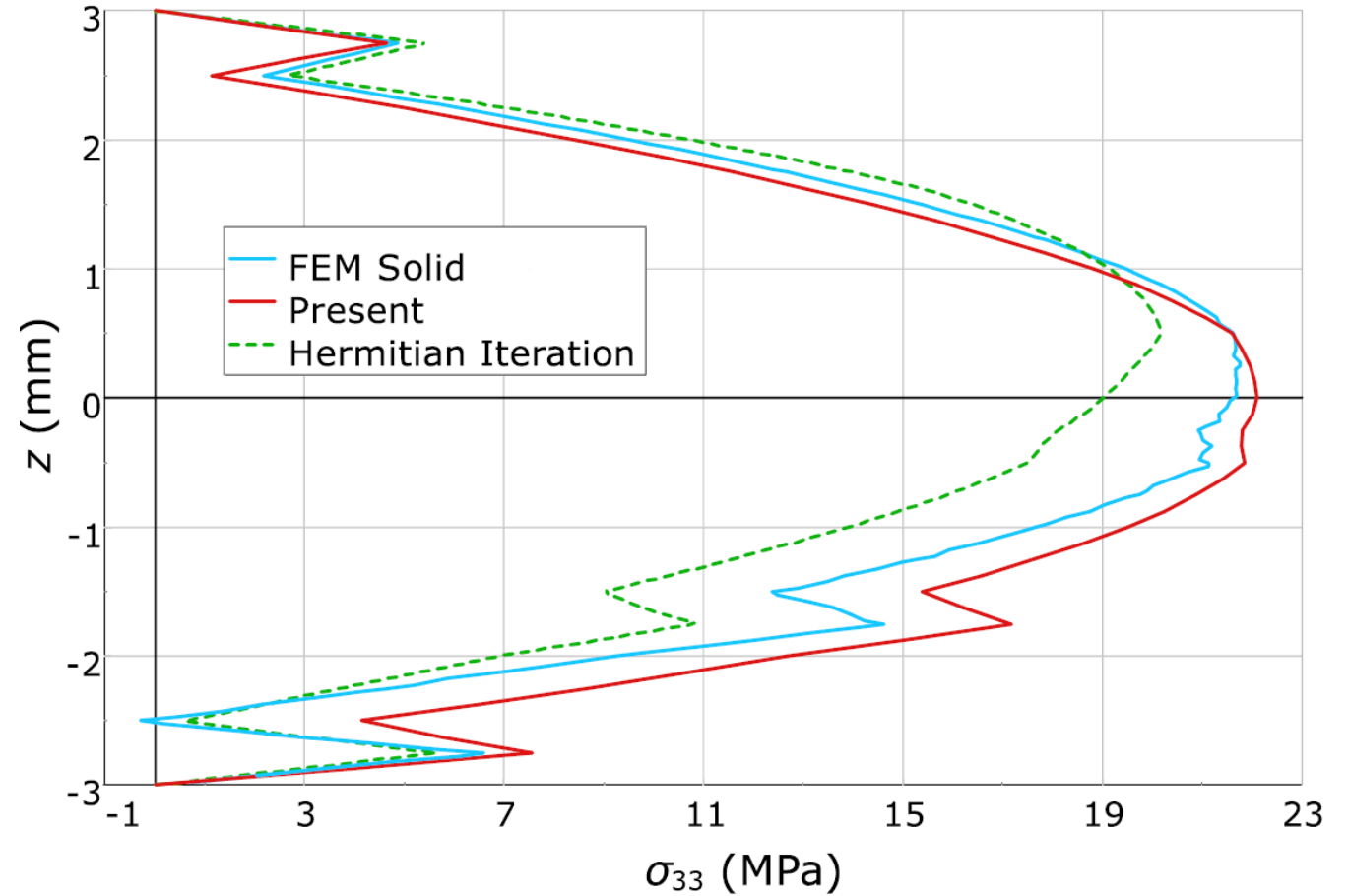
# Doubly-curved example: Bending



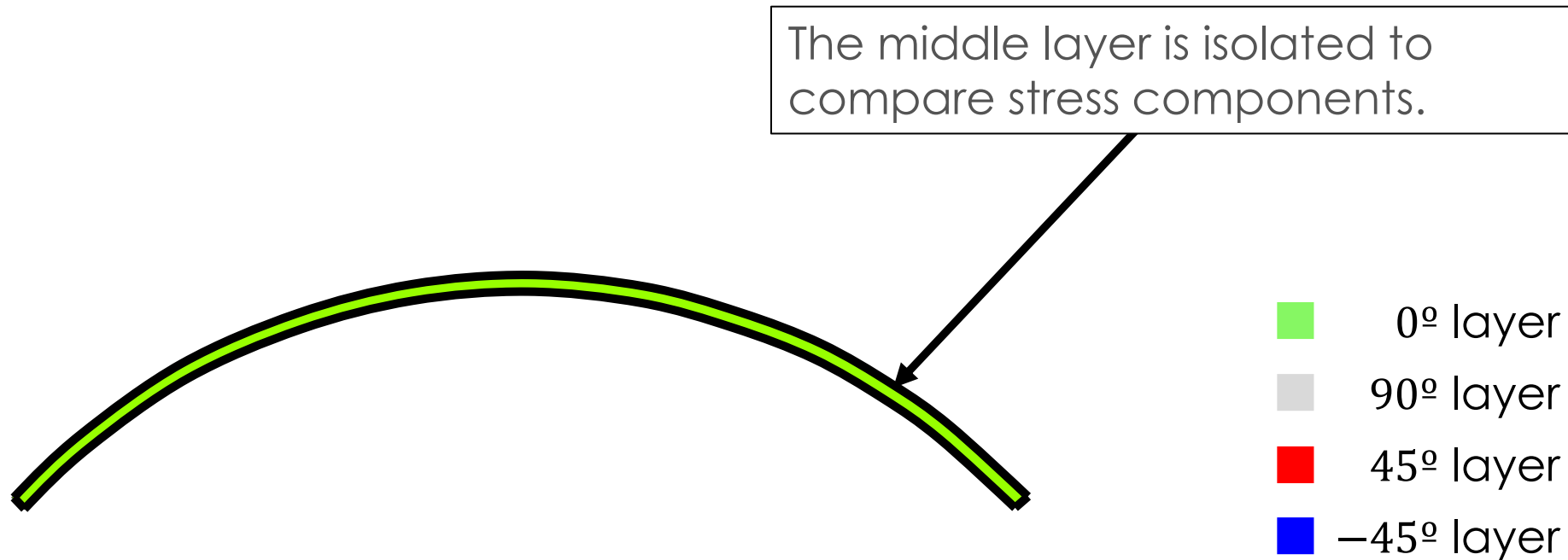
# Doubly-curved example: Bending



Stress 33



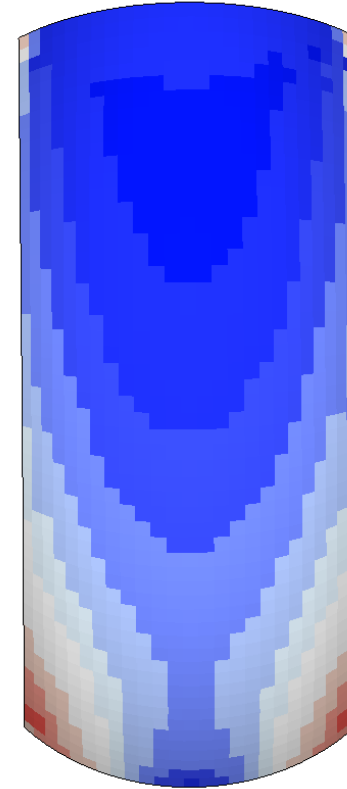
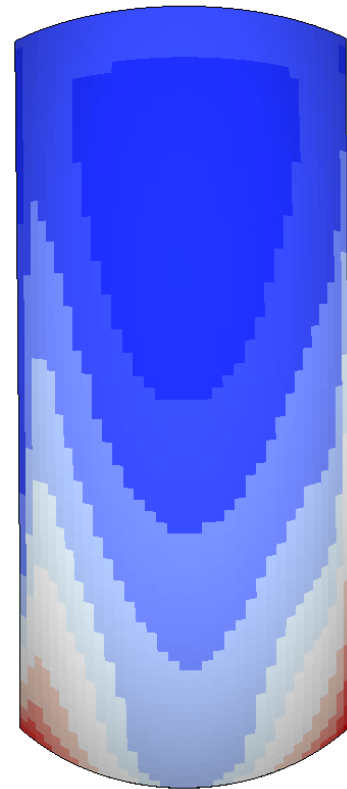
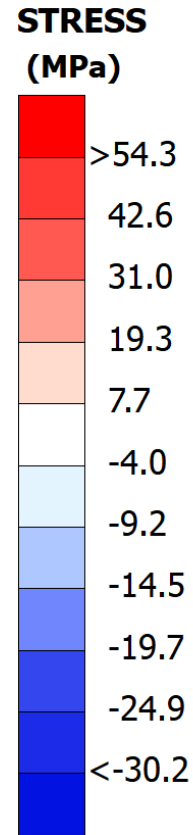
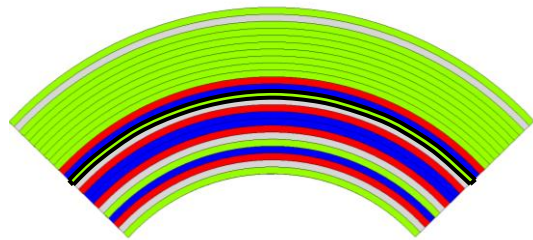
# Doubly-curved example: Bending



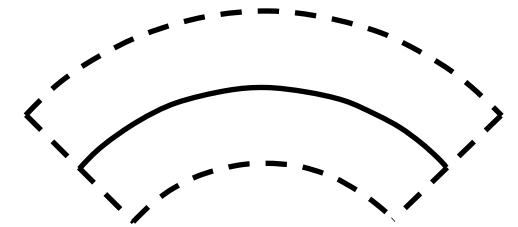
# Doubly-curved example: Bending

Stress 13

Solid



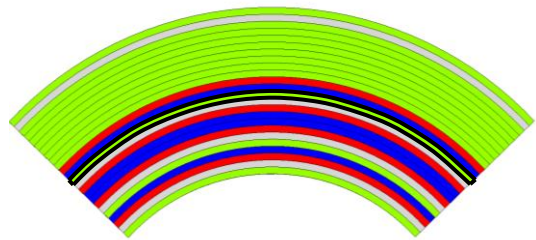
Shell



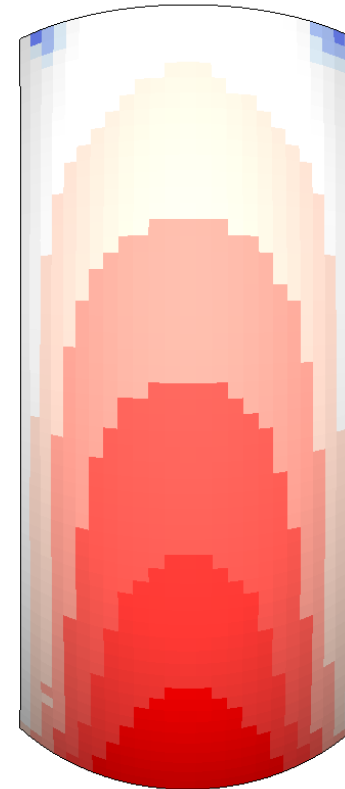
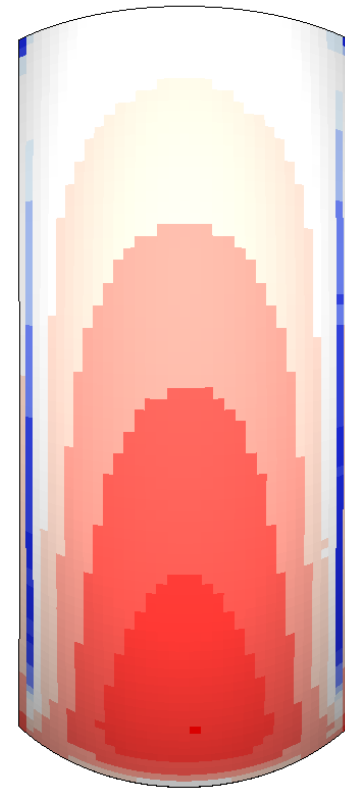
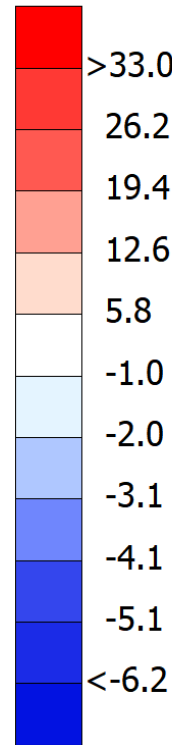
# Doubly-curved example: Bending

Stress 33

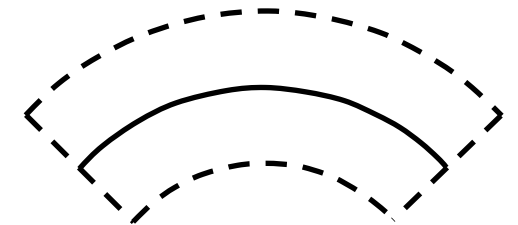
Solid



**STRESS**  
(MPa)



Shell



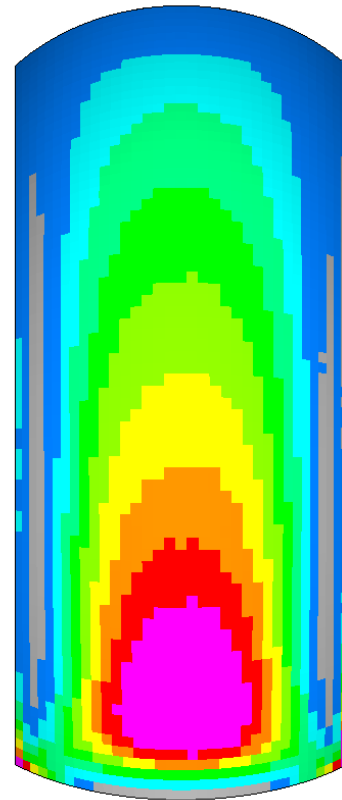
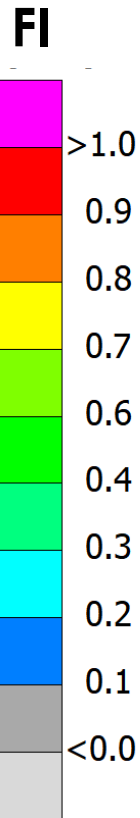
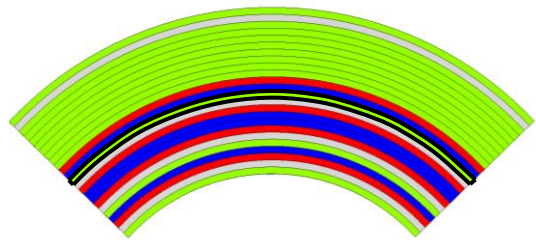
# Doubly-curved example: Bending

## Delamination Criterion

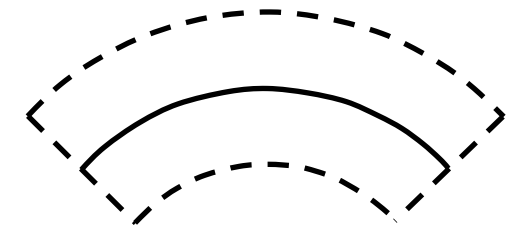
$$\frac{\langle \sigma_{33} \rangle^2}{X_I^2} + \frac{\sigma_{13}^2 + \sigma_{23}^2}{X_{II}^2}$$

$X_I = 30 \text{ MPa}$   
 $X_{II} = 60 \text{ MPa}$

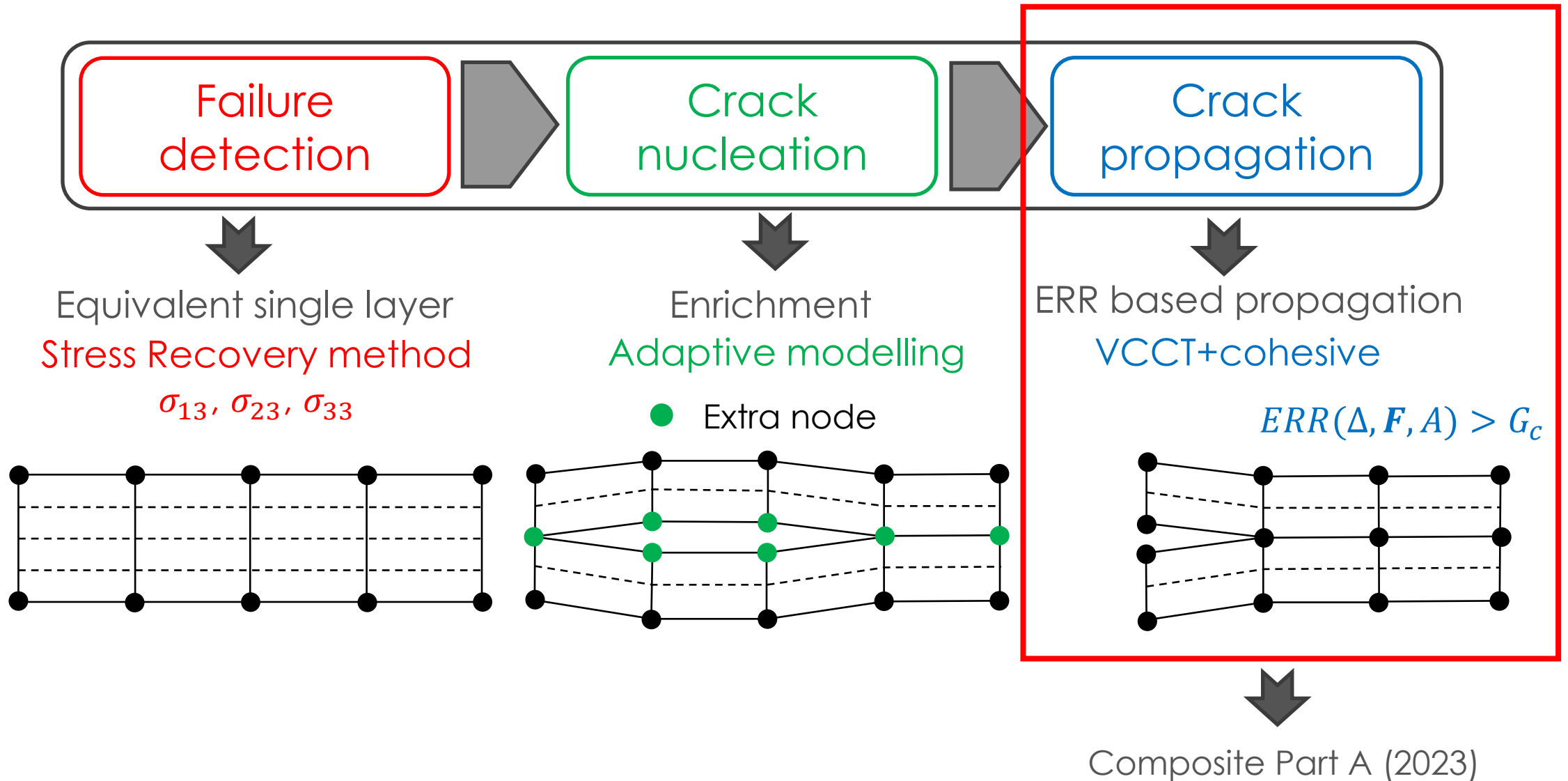
Solid



Shell







# VCCT criterion

- An initial crack is present: will it propagate?
- The Energy Release Rate (ERR) is computed:

$$G_I = \frac{1}{2A^i} F_I^i \langle \Delta_I^{i-1} \rangle \quad ; \quad G_{II} = \frac{1}{2A^i} F_{II}^i \Delta_{II}^{i-1}.$$

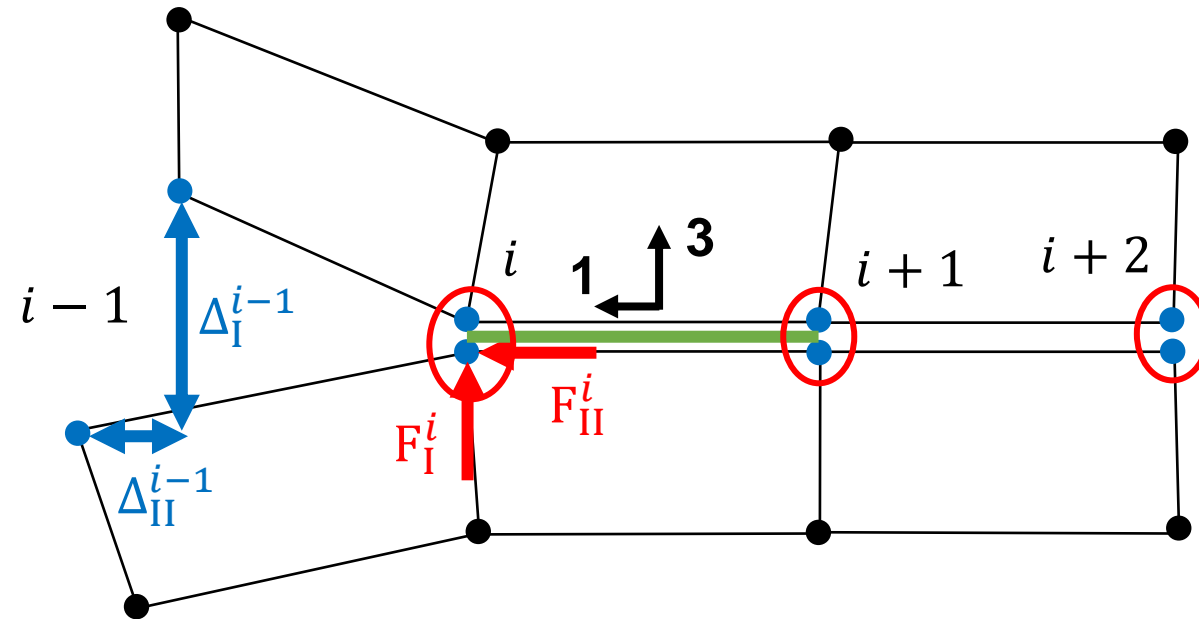
$$G_T = G_I + G_{II}.$$

if  $G_T > G_C$  (Benzeggagh-Kenane)



The tied at node  $i$  is released.

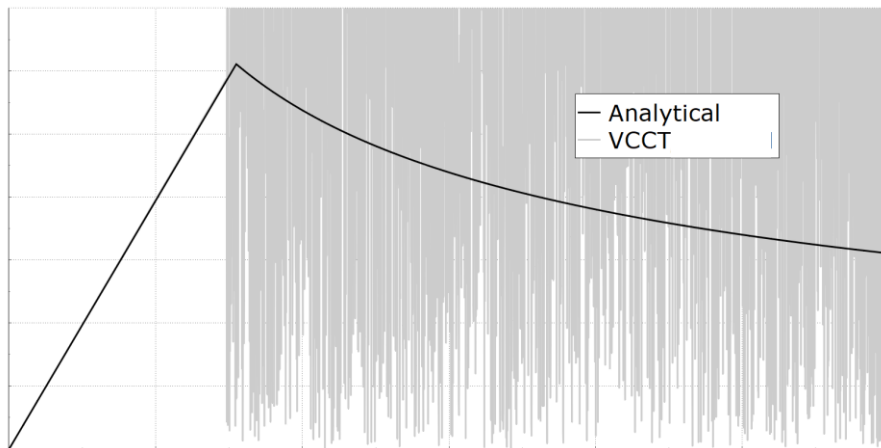
McElroy(2016). NASA/TP-2016-219211.



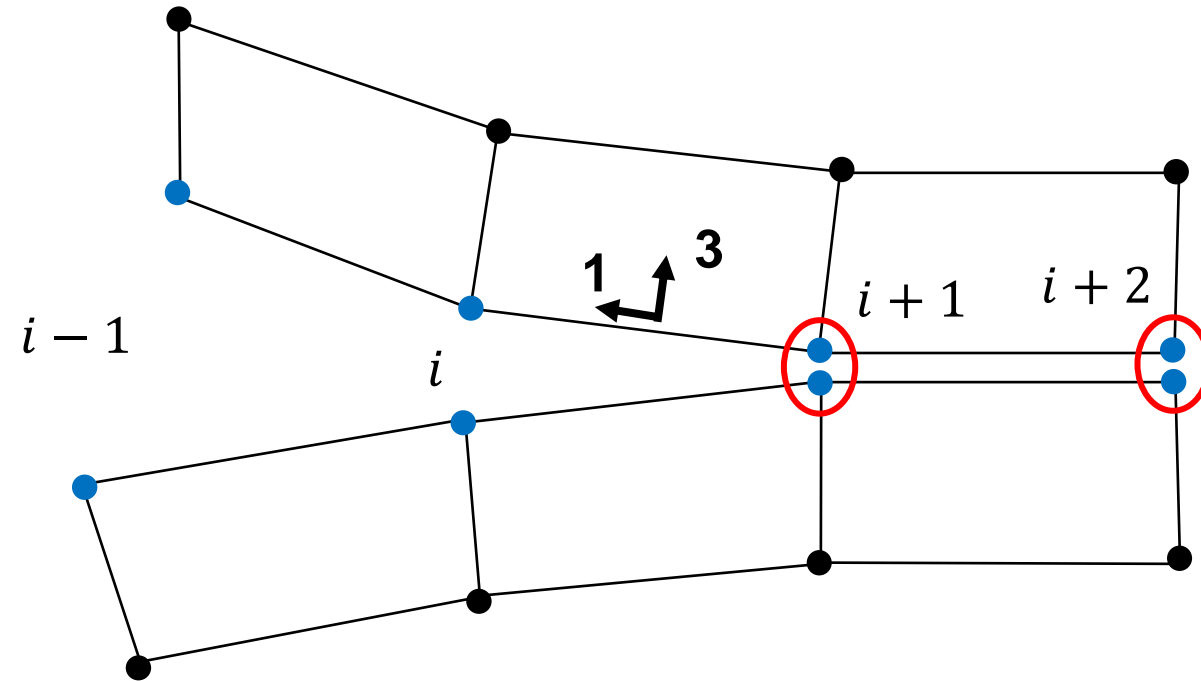
- Base node
- Extra node
- Coincident tied node pair
- Area  $A^i$  to be opened

# VCCT criterion

■ An initial crack is present: will it propagate?



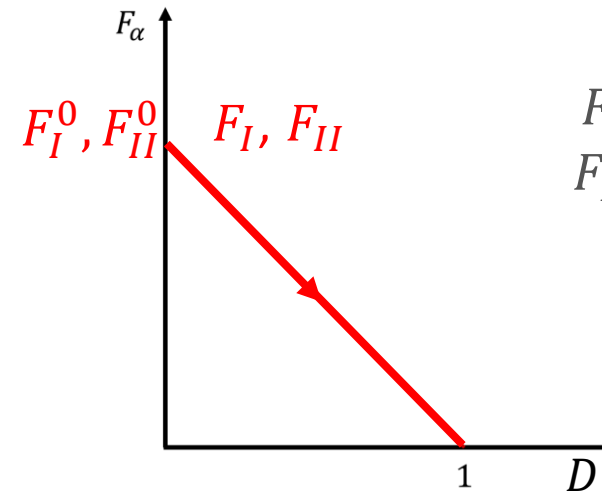
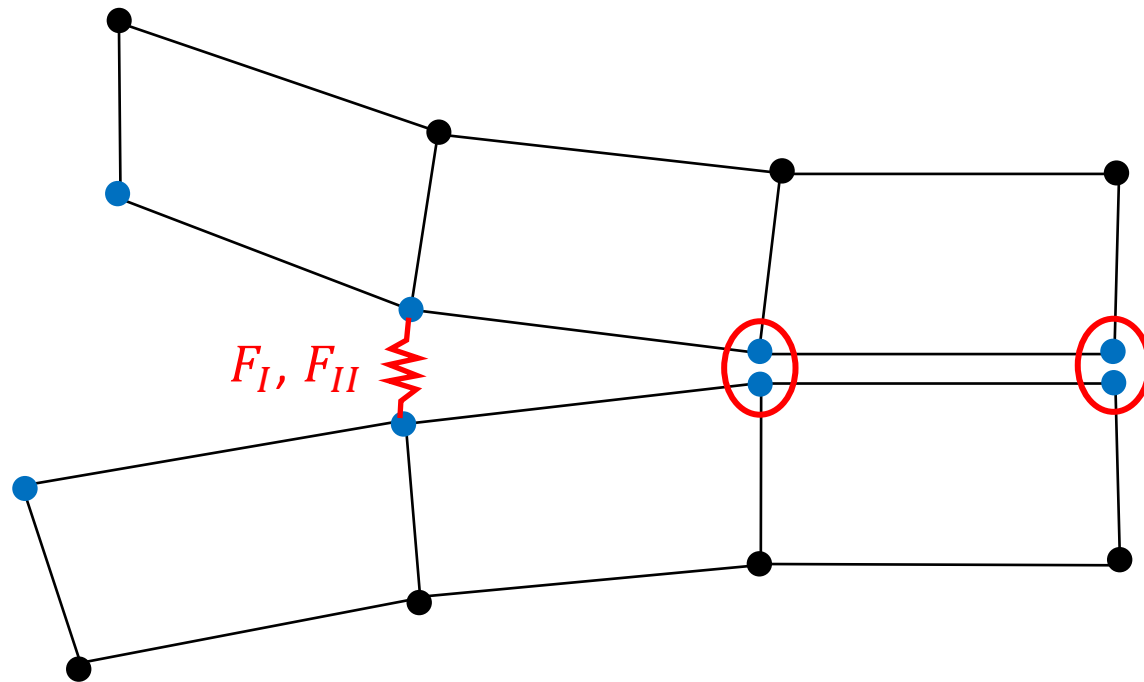
An energy absorption mechanism is necessary !



- Base node
- Extra node
- Coincident tied node pair
- Area  $A^i$  to be opened

# Cohesive law

When the VCCT criterion is met the force at the interface decreases:



$$F_I = F_I^0(1 - D)$$

$$F_{II} = F_{II}^0(1 - D)$$

The damage variable  $D$  ensures that the energy  $G_c A$  is dissipated

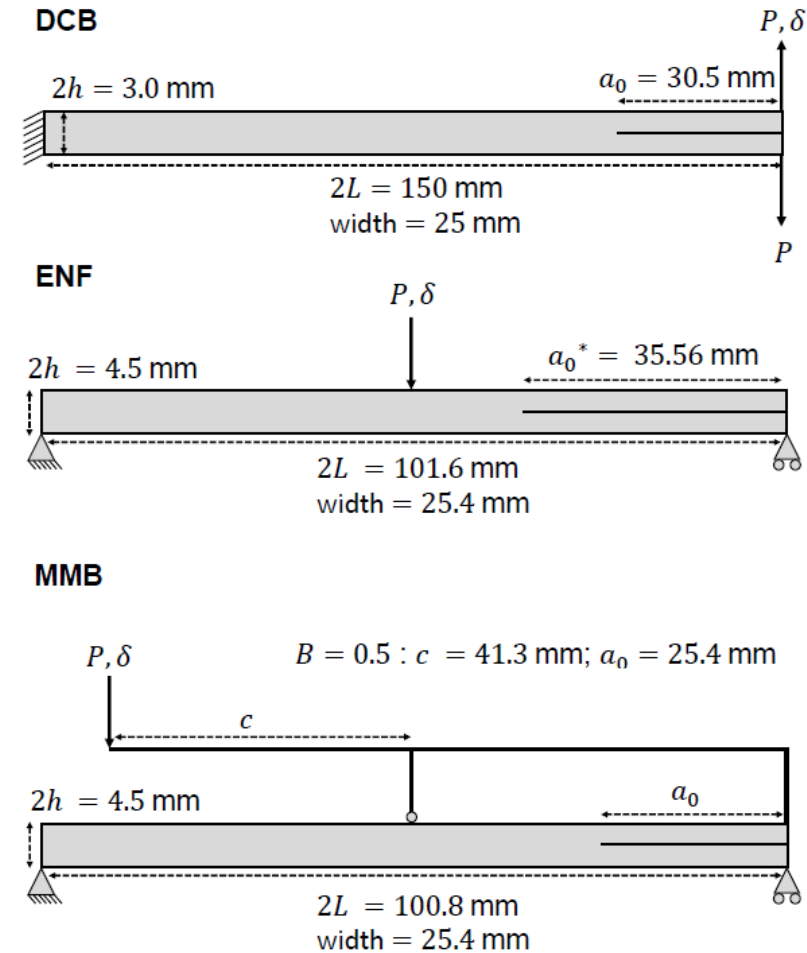
# Validation: DCB, ENF, MMB

- ▣ Benchmark dimension and material from:

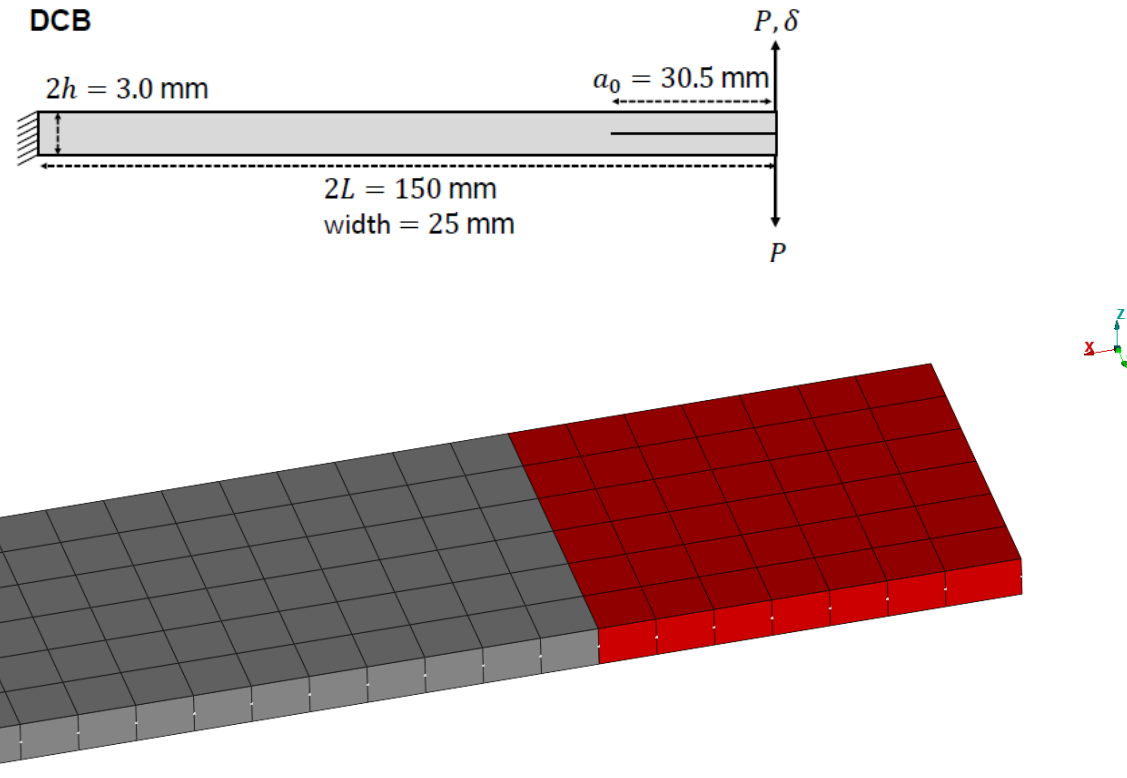
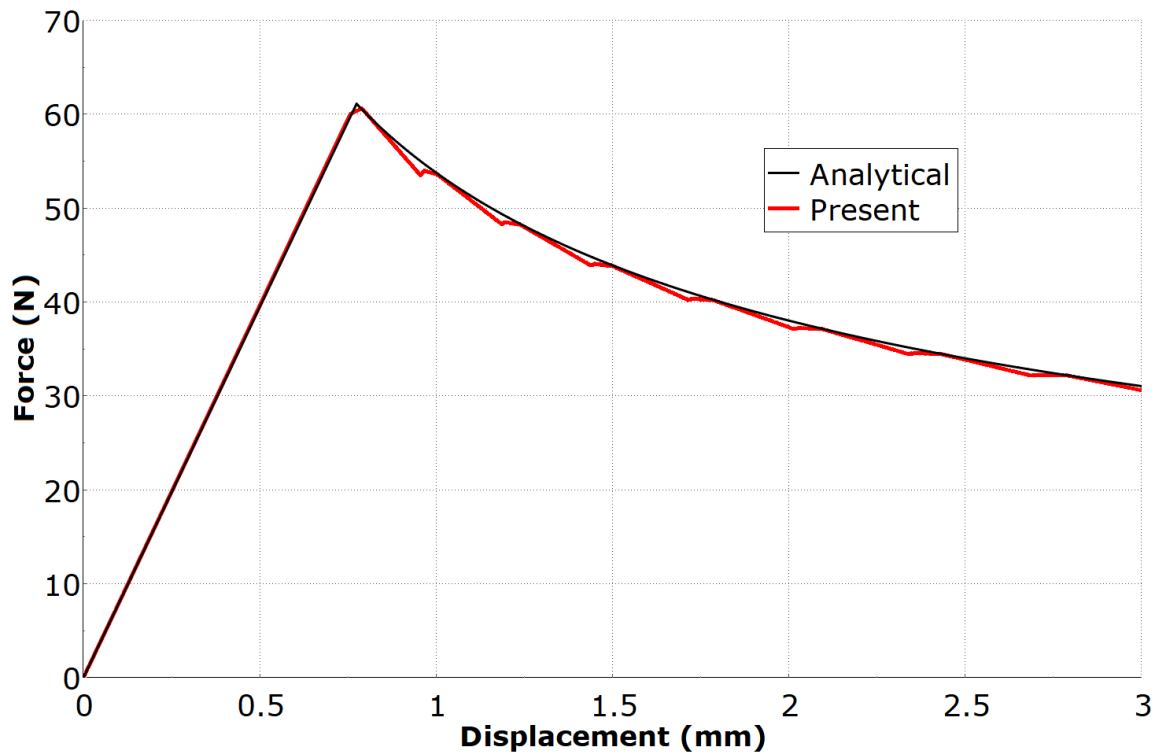
Krueger (2015), J. Compos. Mater.

- ▣ Material Fracture Process Zone  $\approx 0.80$  mm

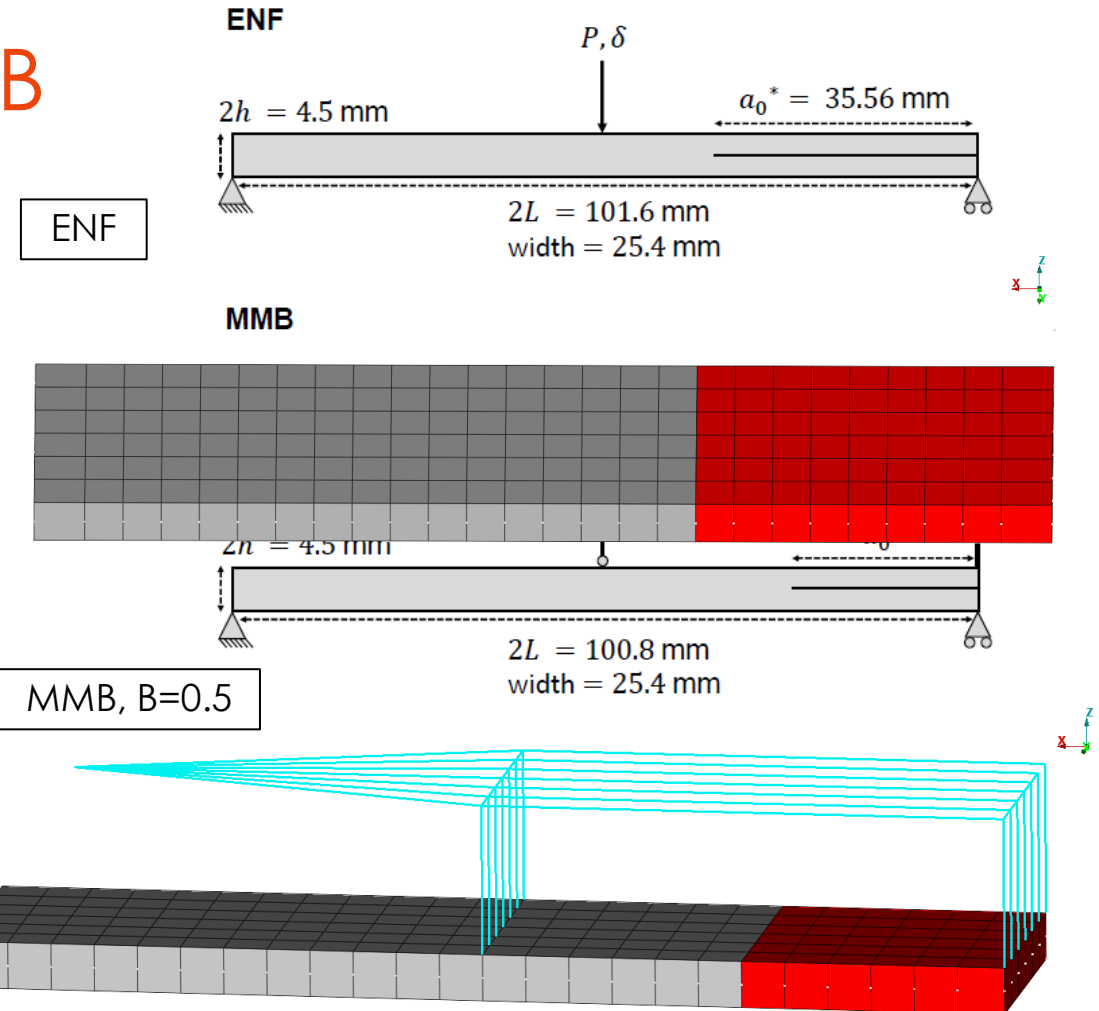
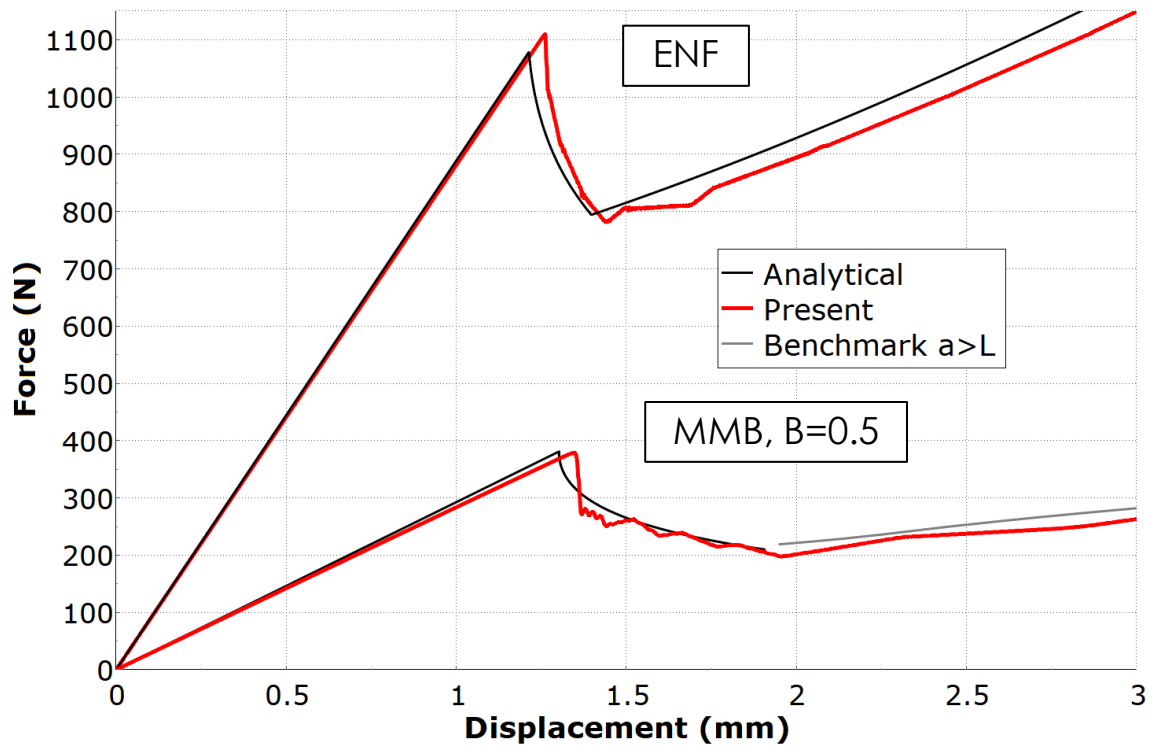
- ▣ Element length: 4 mm



# Validation: DCB, ENF, MMB

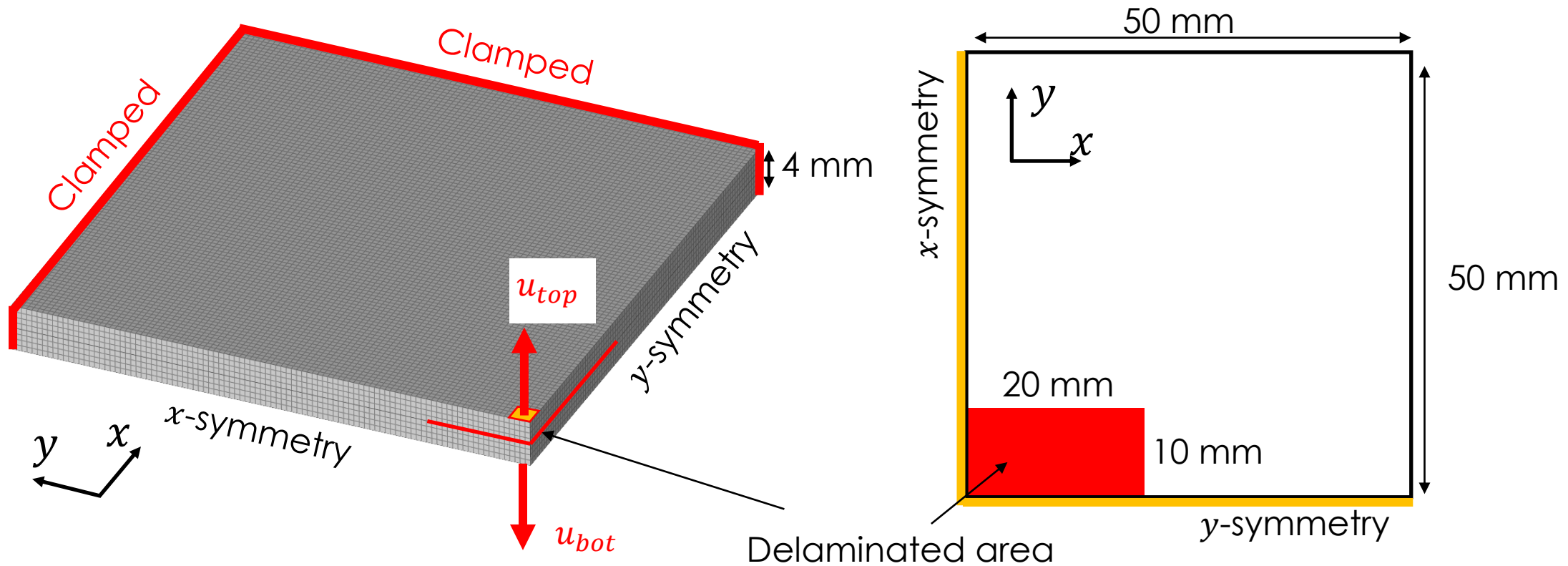


# Validation: DCB, ENF, MMB



# Crack propagation

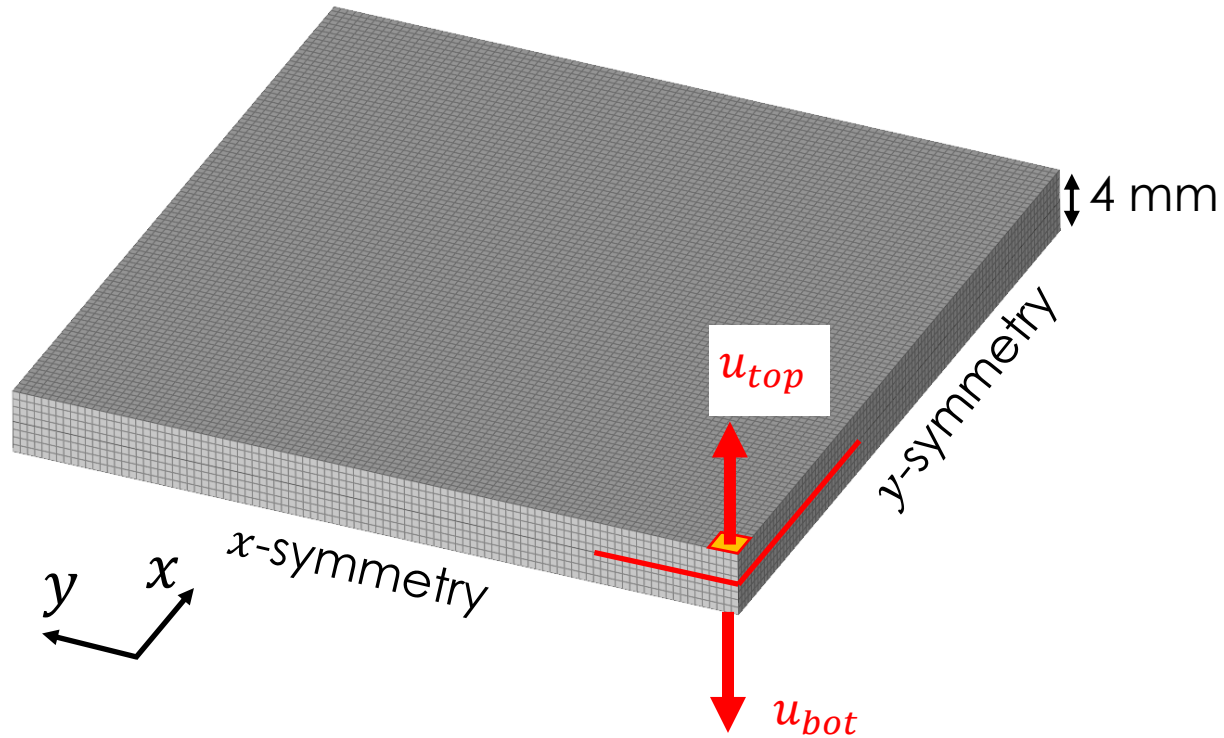
Case from: De Carvalho (2022), NASA/TM-20220002081





# Crack propagation

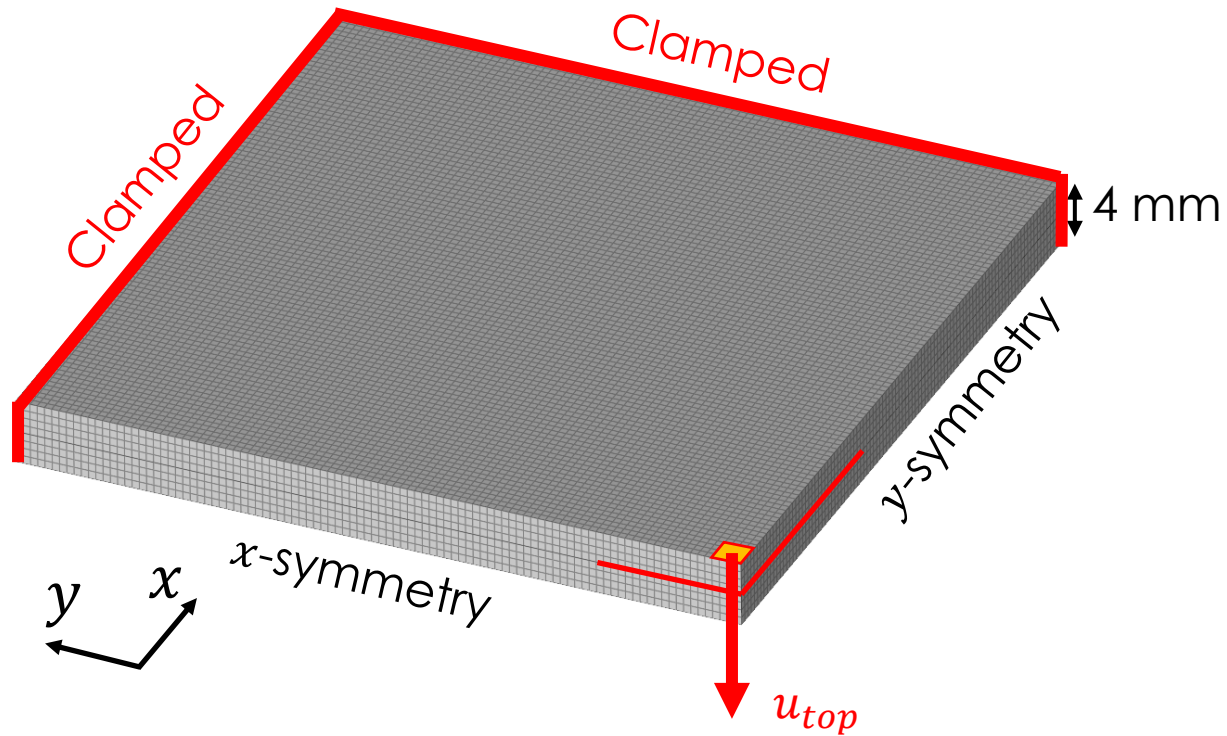
Case from: De Carvalho (2022), NASA/TM-20220002081



Case	$u_{top}$	$u_{bot}$	Edges
Mode I	positive	négative	free
Mode II	negative	-	clamped

# Crack propagation

Case from: De Carvalho (2022), NASA/TM-20220002081



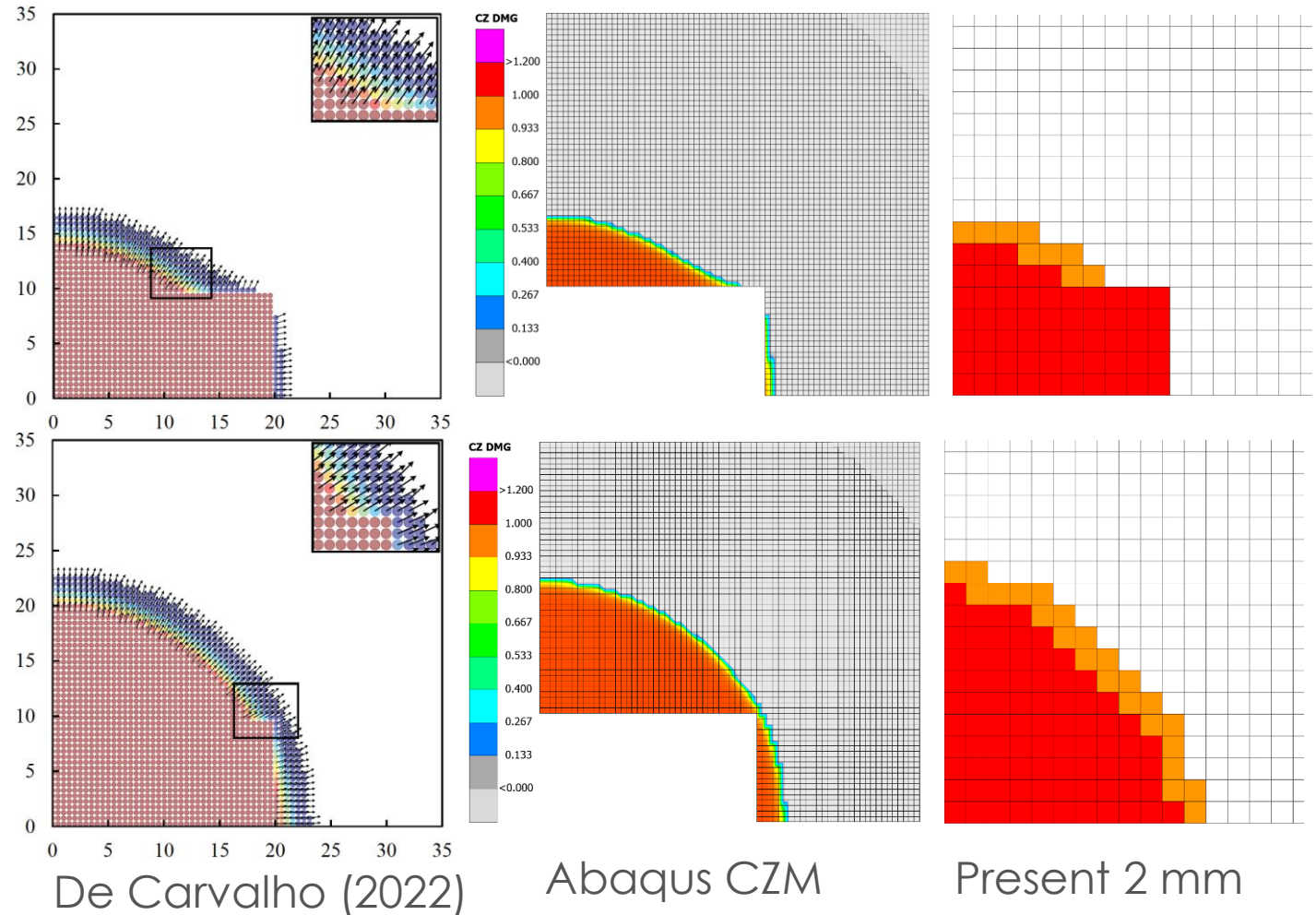
Case	$u_{top}$	$u_{bot}$	Edges
Mode I	positive	négative	free
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# Crack propagation

Mode I

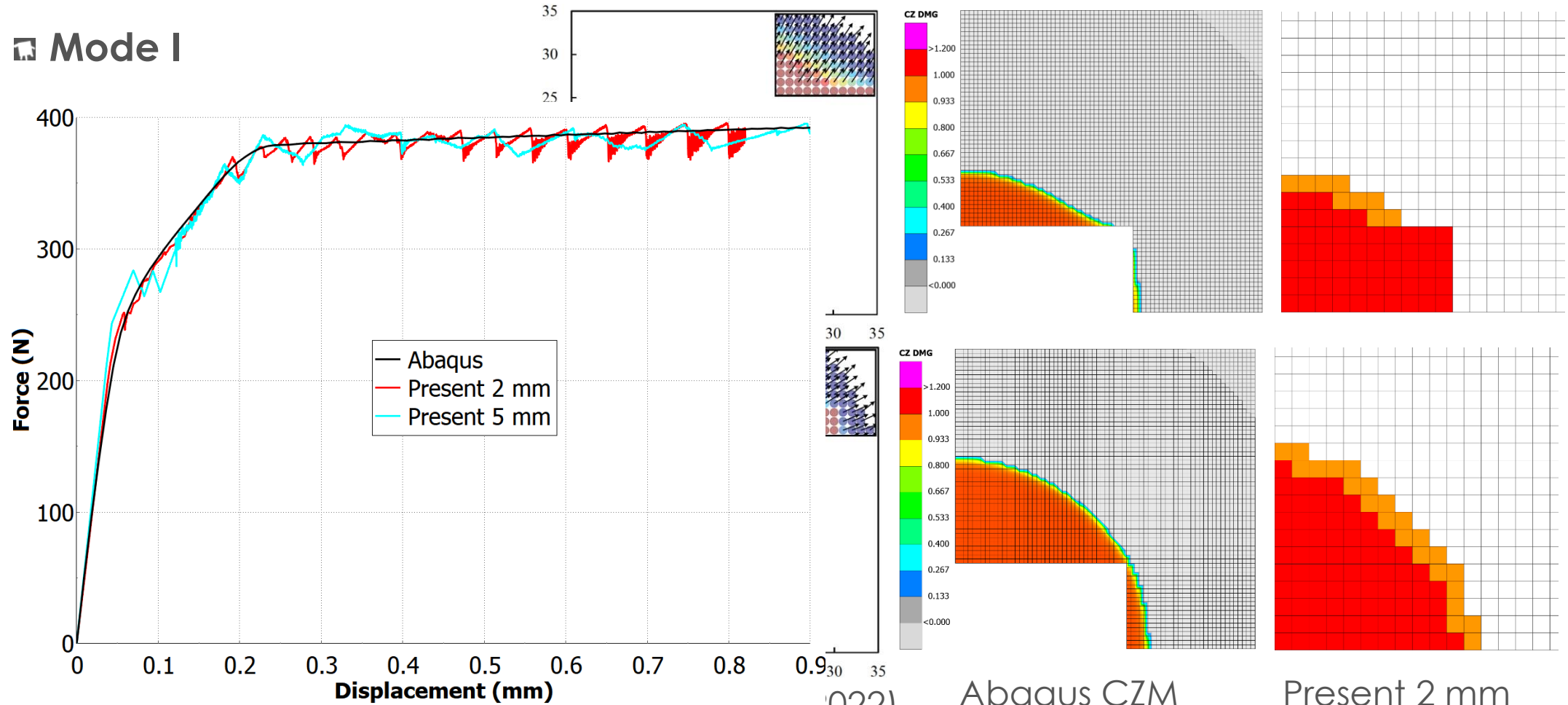
$u_{top} = 0.1 \text{ mm}$

$u_{top} = 0.2 \text{ mm}$



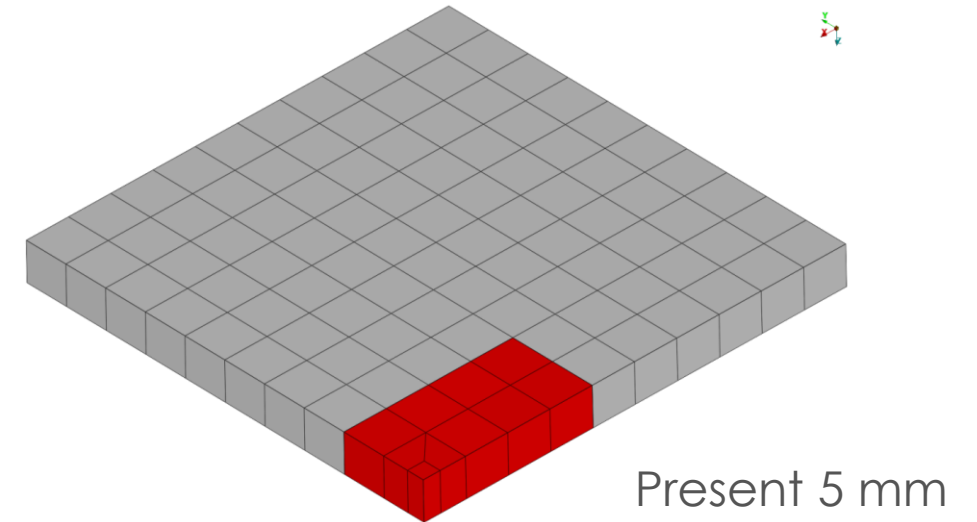
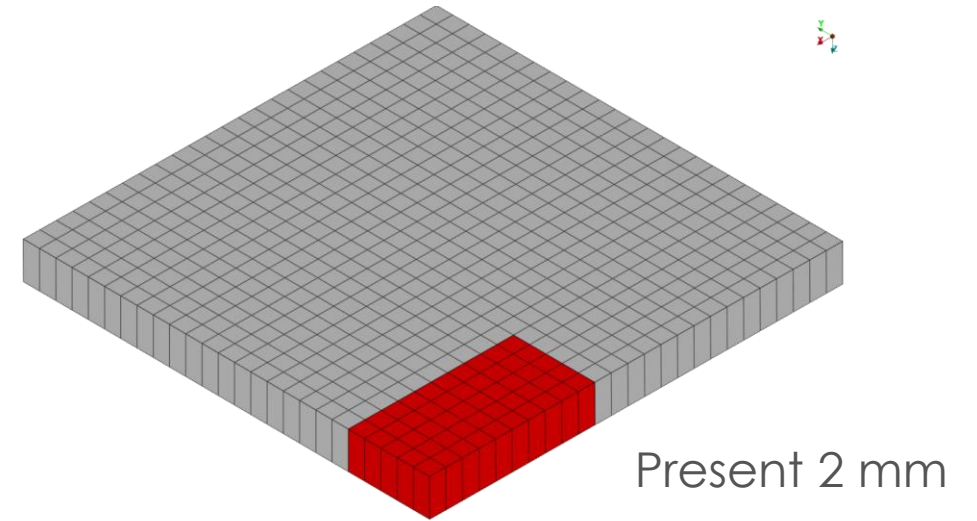
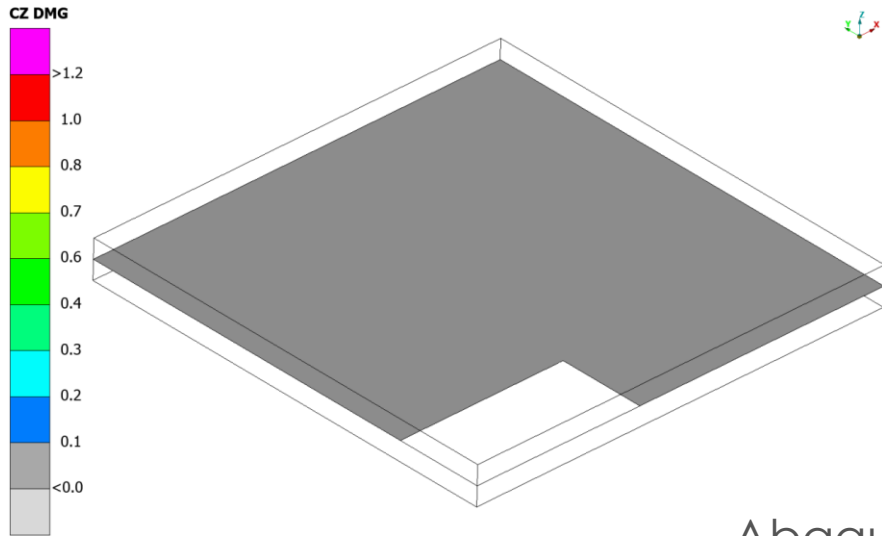
# Crack propagation

## Mode I



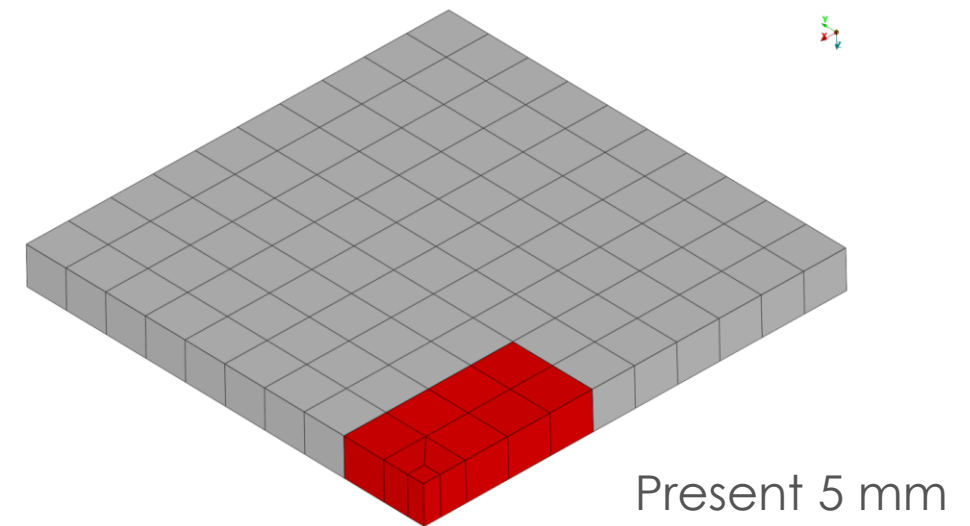
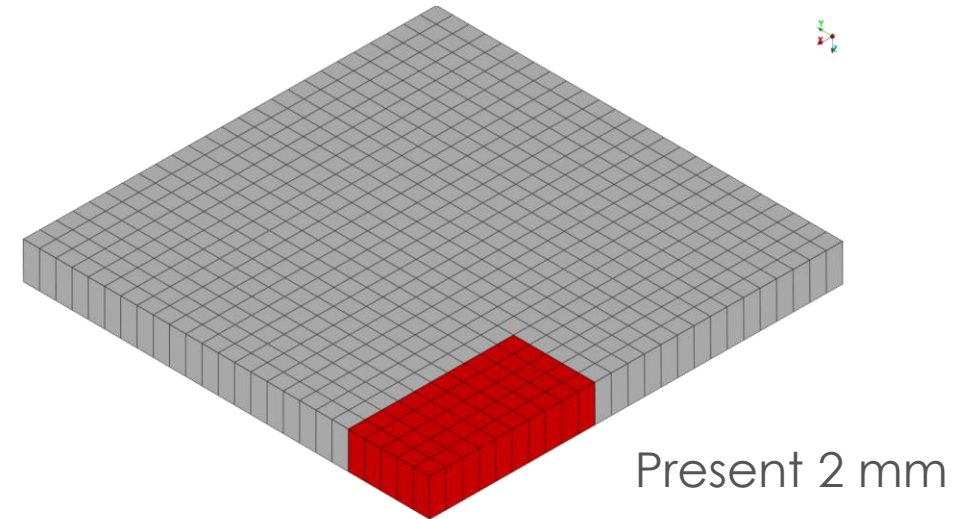
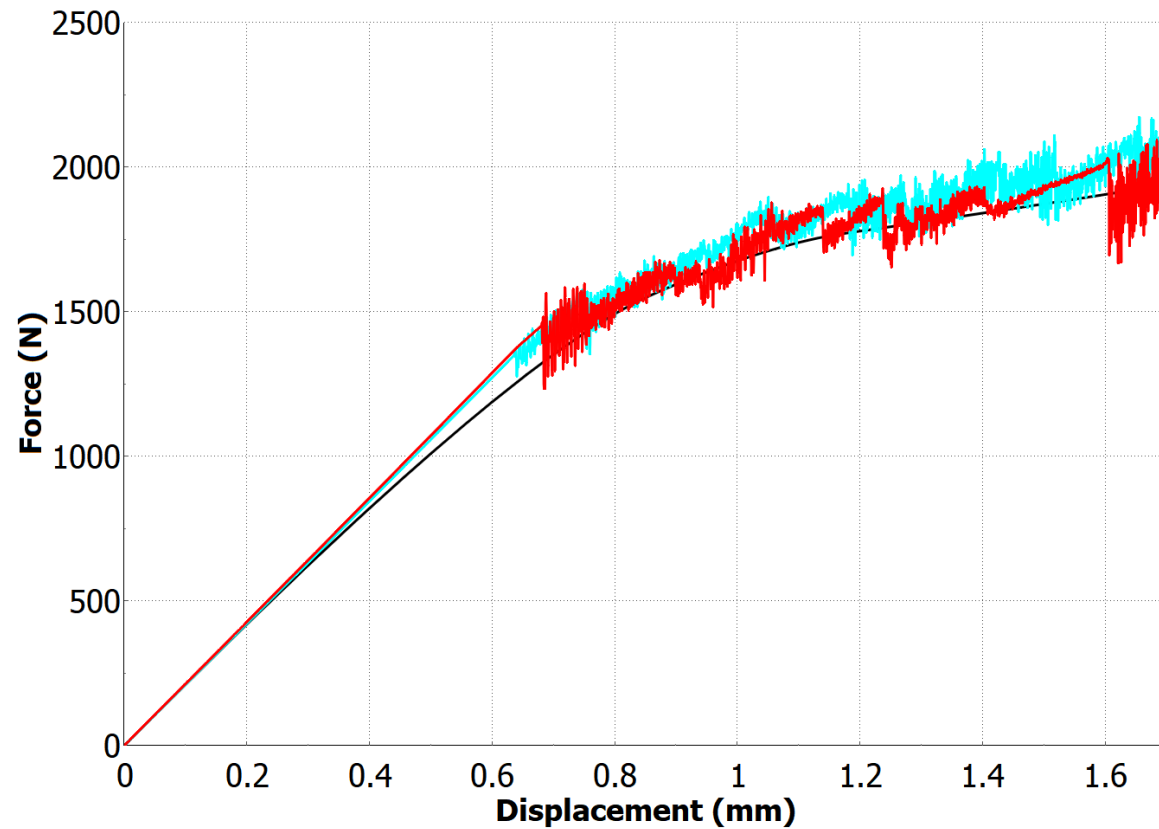
# Crack propagation

Mode II



# Crack propagation

## Mode II



# Conclusions

## Stress Recovery (Delamination initiation)

- ▣ Accurate out-of-plane stresses can be obtained with ESL linear shells.
- ▣ The model can be refined where there is a risk of delamination.
- ▣ Can also be used as a post-processing tool.

## VCCT-Cohesive approach (Delamination propagation)

- ▣ The method is able to model crack propagation in large elements.
- ▣ Does not introduce artificial stiffness.

Using these two methods,  
delaminations can be modelled in large structures !

# Project outlook

ACCIÓ grant (INNOTEC)

- ▣ Collaboration Btech – AMADE
- ▣ End of project: July 2024

Collaboration with Chalmers continues in the frame of the thesis.





Thank you!

<http://amade.udg.edu>